



United States Department of the Interior

BUREAU OF OCEAN ENERGY  
MANAGEMENT, REGULATION, AND ENFORCEMENT

## Buoyancy Casing Analysis

REPORT

Rev. 0

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## Quality Assurance

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The scope of this report is limited to the matters expressly covered. This report is prepared exclusively for and at the request of the BOEMRE. In preparing this report, Keystone Engineering Inc has relied on information provided by the BOEMRE. KEI has made no independent investigation as to the accuracy of the information provided and has assumed the information is accurate and complete.

All findings and conclusions stated in this report are based upon the information as it existed at the time of the report. Changes to the information may affect the findings and conclusions.

In the course of this investigation, KEI had conversations with persons not on the investigation team and this report draws upon the members understanding of those conversations. The investigation team did not record or produce verbatim transcripts of those conversations nor did the team ask the other parties to review or endorse the notes taken by the team members.

Graphics are used to depict information and conditions; these may be simplified or not to scale. They are intended as an aid to the reader in the context of the report.

The view, opinions, and/or findings contained in this report are those of the author(s) and should not be construed as an official Government position, policy or decision, unless so designated by other documentation.



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## Section 1 - Introduction

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### Objective

Keystone Engineering Inc (KEI) was contracted by the Bureau of Ocean Energy Management, Regulation and Enforcement (BOEMRE) to review the design of the BP Macondo Well 9-7/8" by 7" production casing and its hanger and to calculate the static forces acting upon the casing to determine if they could lift the casing and hanger from its design location. The deliverables are:

- Calculation of buoyancy force acting on the 9-7/8" by 7" production casing
- Calculation of forces on the
  - Bottom of the 7" casing
  - Crossover of the 9-7/8" by 7" casing
  - 9-7/8" by 7" casing hanger

The BOEMRE provided KEI with data that included a wellbore schematic, a casing crossover schematic, reports from cement service providers and drilling reports. The data is listed in the Appendix B.

### Method

A spreadsheet model of the wellbore was developed which considered the properties of the annulus casing, production casing, fluids in both the annulus and production casing, and formation temperatures at wellbore depths. This data was derived from the information provided by BOEMRE.

Several possible operating conditions were considered and eight were chosen for detailed calculations. The conditions are:

- Condition 1 - Wellbore as designed with the cement complete as designed. The design includes 14 ppg mud in the annulus up to the seal/hanger at the 5061 foot depth and the production casing up to the Rig Kelly Bushing (RKB). (See Fig 3.1.)
- Condition 2 - Communication through the cement to the annulus that allows the pressure at the bottom of the annulus to drop to the reservoir pressure. The annulus is still full of 14 ppg mud up to the seal at the 5061 feet. Production casing is full of 14 ppg mud. (See Fig 3.2.)
- Condition 3 - Communication through the cement to the annulus that allows production fluid into the annulus. Annulus is filled with production fluid up to the seal at the 5061 feet. Production casing is full of 14 ppg mud. (See Fig 3.3.)
- Condition 4 - Communication through the cement to the annulus that allows production fluid into the annulus. Annulus is filled with production fluid up to the seal at the 5061 feet. The production casing is filled with gel, seawater and mud. The mud level is at 8367 feet, which is the depth of the drill string in the wellbore schematic. (See Fig 3.4.)
- Condition 5 - Communication through the cement that allows production fluid into the production casing. Annulus is filled with 14 ppg mud up to the seal at the 5061 feet. The production casing is filled with gel, seawater and mud. The mud level is at 8367 feet which is the depth of the drill string in the wellbore schematic. (See Fig 3.5.)
- Condition 6 - Communication through the cement that allows production fluid into the production casing. Annulus is filled with production fluid up to the seal at the 5061 feet. The production casing is filled with static production fluid. (See Fig 3.6.)



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- Condition 7 - Communication through the cement that allows production fluid into the production casing. Annulus is filled with 14 ppg mud up to the seal at 5061 feet. The production casing is filled with static production fluid. (See Fig 3.7.)
- Condition 8 - Communication through the cement that allows production fluid into the production casing. Annulus is filled with production fluid up to the seal at 5061 feet. The production casing is filled with static production fluid. The pressure in the casing at 5061feet is set to 2358 psi, which is the pressure from Condition 5 with gel and seawater in the casing above the wellhead. This is a close approximation to the pressure if the production fluid is flowing to the surface. The production fluid is not modeled in HSYS since the flow rate is not known. This condition will give an estimate of the highest pressure in the annulus and the lowest pressure in the casing. (See Figure 3.8)

These conditions are detailed in Section 3 of this report. Diagrams showing the fluid levels and the pressures are included for each condition.

The production casing was considered to have been set on its hanger while the cement was still fluid so the weight of the casing is supported by the hanger only and no axial support is given by the cement or the bottom of the hole. Without this assumption, the forces on the hanger cannot be determined.

The buoyant forces in each condition are developed using the principle of Archimedes that the buoyant force on a submerged object is the weight of the displaced fluid. The force varies depending on the weight of the fluid in the annulus; in this study the fluid is either 14 ppg mud or production fluid from the reservoir. The shape of the object has no effect on the buoyant force, so the pressure differentials at the 9-7/8" by 7" crossover have no effect.

The differential pressure between the annulus and the interior of the production casing will cause the casing dimensions to change from the nominal design. The amount of deformation was calculated to determine the effect. The temperature of the casing will also change the dimensions, and those changes were determined. The calculations are included in Section 4.

The production casing is made of Q-125 steel (9-7/8") and HC-125 steel (7"). These steels are classified as Low Chrome Moly steels. The properties of Chrome Moly steels were taken from the table for "Low Chrome Moly steels, chrome < 3%" from ANSI B31.3. The properties are included in Section 4.

The dimensions of the 9-7/8" by 7" crossover were taken from the drawing by K&B (Appendix B). The weight and volumes of the crossover were calculated separately and entered into the buoyancy calculations as a combined weight and volume. The details are included in Section 4.

Differences in the annular and interior pressures at the crossover will result in net radial forces at the crossover. Also, due to the shape of the crossover (larger top reducing to a smaller bottom), the areas exposed to the annular and interior pressures differ. The crossover is therefore subject to a net vertical force that is calculated in Section 4. Note that this vertical force is included in the buoyancy force and is not additional. The principle of Archimedes uses displacement weights, the shape of the object (hull) is not considered.



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The temperature and compressibility properties of the Synthetic Oil Based Mud (SOBM) were not available so a constant 14 ppg density was used.

The density of the production fluid was developed by entering the component analysis from Schlumberger (Appendix B, Item 6) into a HYSYS model for a static column from the bottom of the well to the hanger. The boundary conditions for the column are the bottom hole temperature and pressure and an upper temperature of 110 Deg F at the rig floor. The upper temperature was chosen to match wellhead temperatures commonly observed in production operations. The HYSYS model is detailed in Section 5.

The capacity of the casing strings is calculated in Section 4.

The forces on the bottom of the casing were determined from the following condition. The production casing and its annulus were filled with mud and the cement was pumped into the bottom of the well with the casing elevated above the hanger. The casing was then lowered into the cement, which was still fluid. The casing was then supported by the hanger. The pressures on the annulus and interior sides are equal, and the axial force is that pressure times the area of the 7" 32 lb/ft casing. The axial force is then the hydrostatic pressure (13,200 psi) times the casing metal area (10 sq in). Three conditions are possible:

- The cement seals the casing interior but allows communication between the annulus and the reservoir. The axial force remains the same, 132,000 lb.
- The cement allows communication between the interior and the reservoir but seals the annulus. The hydrostatic force is the reservoir pressure.
- The reservoir communicates with both the annulus and the interior. The hydrostatic force is the reservoir pressure.

The resulting forces and pressures are listed in the results for each condition. Note that these forces are included in the buoyant force calculation and are not to be added to the buoyant force.

Temperatures along the casing string will change with fluid flow from the reservoir. The casing was assumed to be in equilibrium with the formation temperatures when the casing is set. Determination of casing temperatures depends on many factors (e.g. fluid properties, flow rates, cement and annular fluid properties, reservoir properties and time) that are not known. The determination of forces due to temperature expansion in the casing is beyond the scope of the investigation.



## Section 2 - Executive Summary

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The design of the 9-7/8" by 7" production casing for the Macondo well was reviewed for determination of the buoyant force that would lift the production casing from its hanger. Eight possible operating conditions of the wellbore were examined. The key results are listed below:

- Conditions 5 and 8 result in a net upward force on the hanger.
- The weight of the production casing steel is greater than the weight of the displaced fluid if the fluid is 14 ppg mud. Since the mud is the heaviest of the displaced fluids, the production casing is not buoyant if the pressure forces at the hanger are not considered.
- The weight of the contents of the production casing opposes the buoyant force if the cement at the bottom of the casing seals the casing. The weight of the contents is the largest force in the calculation.
- The net buoyancy of the production casing is:
  - the weight of the displaced fluid
  - minus the weight of the steel
  - minus the weight of the contents of the casing (if the cement seals the contents)
- If the cement at the bottom of the casing does not seal the casing, the weight of the contents has no influence on the buoyant force.
- The net force on the production casing hanger is:
  - the net buoyancy of the casing
  - plus the annulus pressure at the hanger times the projected annular area of the hanger
  - minus the interior pressure of the casing times the projected inside area of the casing
  - This calculation assumes the production casing is set with no support from the cement or the bottom of the hole, i.e. the casing is set on fluid cement.
- The annulus pressure and pressure inside the production casing at the hanger create large forces.
  - If the cement at the bottom of the casing does not seal, the net buoyancy becomes positive (upwards) if the annulus pressure is greater than the interior pressure by a certain amount. This amount changes for each condition depending upon the density of the fluids in the annulus and interior.
  - If the cement seals, the net buoyancy will be negative (downwards) regardless of annulus pressure.
- The temperatures and pressures of the well do not change the radial dimensions of the production casing enough to be significant. The changes are about 0.13% (temperature of 262 Deg F) and 0.18% (differential pressure of 6572 psi).
- The production fluid static column was modeled in HYSYS using reference temperatures of 110 and 160 Deg F to determine sensitivity to the assumed temperature. The net change was 0.36% which is considered negligible.
- Forces due to temperature changes of the production casing were not determined.



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### Condition 1

#### *Basis*

The wellbore is filled with mud fluid and the cement is intact as designed.

The production casing is filled with mud to the Rig Kelly Bushing (RKB).

The annulus is filled with mud to the production casing hanger.

Cement seals the reservoir from the annulus and the production casing, so the pressures in the well are hydrostatic pressures of the mud.

Since the production casing is sealed at the bottom, lifting the casing will lift the contents of the casing up to the RKB, and so the weight of the contents is included in the buoyancy calculation. This weight of the contents will act to oppose the buoyancy force.

The hydrostatic pressure in the annulus at the bottom of the hanger is due to mud fixed in the annulus after cementing when the production casing is lowered into the hanger. This annulus pressure acts upon the exposed annular area of the production casing hanger creating an upward force that adds the buoyant force. This upward force is opposed by the hydrostatic pressure in the production casing acting on the exposed interior area of the hanger.

The area of the production casing hanger exposed to the annulus pressure is the area between the inner diameter of the 18-3/4 inches casing (18.62 inches) and the outer diameter of the production casing hanger (9-7/8 inches). This area was determined to be 195.7 sq. in. (from the Dril-Quip drawing 2-PD-32424-02CP, Appendix B, Item 3).

The area of the production casing hanger exposed to the production casing pressure is the area between the inner diameter of the 18-3/4" hanger (18.62 inches) and the inner diameter of the production casing hanger (8.553 inches). This area was calculated to be 214.8 sq. in. (from the Dril-Quip drawing 2-PD-32424-02CP, Appendix B, Item 3).

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or (annular or interior) fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the cement to hold the production casing in place was not considered.

The shoe and bottom of the hole formation will resist the lowering of the production casing. Since this effect is applicable only if the casing is lowered by a negative buoyant force and this study is considering the lifting of the casing, the resistance of the shoe and formation is outside the scope of this study.

The buoyancy calculation sheet (Table 3.1) and Figure 3.1 are included in this section.



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Area calculations are included in Section 4.

### Results

- The buoyancy is a negative 1.63 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 1.70 million lb, so the hanger will not lift
- The resultant of the pressure forces on the hanger is a downward force.
- The casing and hanger will not lift in this condition.
- The weight of the production casing is more than the weight of the displaced mud. Since this condition has the heaviest displaced weight of all the conditions, this indicates that the buoyant force will be positive (cause the casing to lift) only with a high differential between the annulus pressure and the production casing pressure at the hanger.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 9077 psi
  - Interior pressure 9077 psi
  - Vertical force 73500 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 13200 psi
  - Interior pressure 13200 psi
  - Vertical force 132000 lb, upwards

### Condition 2

#### Basis

The wellbore is filled with mud fluid. Communication through the annulus cement allows reservoir fluid into the annulus.

The production casing is filled with mud to the RKB.

The annulus is filled with mud to the production casing hanger and is sealed at the hanger. The pressure at the top of the annulus is determined by hydrostatic weight of the mud using the reservoir pressure as the known pressure. The pressure at the bottom of the annulus is reservoir pressure. Mud flows into the reservoir until pressures are equal.

Cement seals the reservoir from the production casing, so the pressure inside the production casing is the hydrostatic pressure of the mud.

Since the production casing is sealed at the bottom, lifting the casing will lift the contents of the casing up to the RKB, and so the weight of the contents is included in the buoyancy calculation. This weight of the contents will act to oppose the buoyancy.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.



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Friction between the production casing and cement or annular or interior fluids was not considered. The production casing does not lift in this condition so friction does not apply. The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing was ignored per the reasoning stated in Condition 1.

The buoyancy calculation sheet (Table 3.2) and Figure 3.2 are included in this section.

### Results

- The buoyancy is a negative 1.63 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 1.84 million lb, so the hanger will not lift
- The decrease in the net force on the hanger from Condition 1 is due to the reduction in annulus pressure at the hanger.
- The resultant of the pressure forces on the hanger is a downward force.
- The casing and hanger will not lift in this condition.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 8354 psi
  - Interior pressure 9077 psi
  - Vertical force 46078 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 13200 psi
  - Interior pressure 11844 psi
  - Vertical force 132000 lb, upwards

### Condition 3

#### Basis

Communication through the annulus cement allows reservoir fluid into the annulus. The annulus is filled with production fluid whose properties are determined from a HYSYS model for a static column. The composition of the production fluid was taken from the Schlumberger report (Appendix B, Item 6).

The production casing contains mud to the RKB.

This condition would occur if the reservoir – annulus communication occurred before the seawater was introduced into the production casing.

Cement seals the reservoir from the production casing, so the pressure inside the production casing is the hydrostatic pressure of the gel/seawater/mud column.

Since the production casing is sealed at the bottom, lifting the casing will lift the contents of the casing up to the RKB, and so the weight of the contents is included in the buoyancy calculation. This weight of the contents will act to oppose the buoyancy.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.



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The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or annular or interior fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1

The buoyancy calculation sheet (Table 3.3), HYSYS model description and results, and Figure 3.3 are included in this section.

### Results

- The buoyancy is a negative 2.0 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 0.34 million lb, so the hanger will not lift
- The displaced fluid weight is reduced from Condition 1 because the production fluid is lighter than the mud.
- The annulus pressure at the hanger is higher than the interior pressure in the production casing, adding to the buoyant force.
- The net buoyancy force is greater than Condition 1.
- The casing and hanger will not lift in this condition.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 10540 psi
  - Interior pressure 9077 psi
  - Vertical force 128990 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 13200 psi
  - Interior pressure 11835 psi
  - Vertical force 132000 lb, upwards

## Condition 4

### Basis

Communication through the annulus cement allows reservoir fluid into the annulus. The production casing contains a drill string from the RKB to 8367 feet TVD. The production casing contains seawater above 8367 feet through the wellhead to the riser. The riser is filled with seawater to 300 feet below the RKB. The top 300 feet contains a gel with a density of 16.3 ppg.

This condition would occur if the reservoir – annulus communication occurred after the seawater was introduced into the production casing.

The annulus is filled with production fluid with properties per Condition 3.



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Cement seals the reservoir from the production casing, so the pressure inside the production casing is the hydrostatic pressure of the mud.

Since the production casing is sealed at the bottom, lifting the casing will lift the contents of the casing up to the RKB, and so the weight of the contents is included in the buoyancy calculation. This weight of the contents will act to reduce buoyancy.

The drill string is supported by the rig and has no effect on the buoyancy of the production casing.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or (annular or interior) fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1

The buoyancy calculation sheet (Table 3.4), the HYSYS model description and results, and Figure 3.4 are included in this section.

### Results

- The buoyancy is a negative 1.56 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 0.34 million lb, so the hanger will not lift.
- The weight of the production casing contents is reduced since the gel/seawater/mud column is lighter than that of 14 ppg mud.
- The net buoyancy force is greater than Condition 1.
- The annulus pressure at the hanger is greater than in Condition 1.
- The resultant of the pressure forces on the hanger is an upward force.
- The casing and hanger will not lift in this condition.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 10540 psi
  - Interior pressure 6818 psi
  - Vertical force 196363 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 13200 psi
  - Interior pressure 11835 psi
  - Vertical force 132000 lb, upwards



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### Condition 5

#### *Basis*

Communication through the production casing cement allows reservoir fluid into the production casing. The production casing contains a drill string from the RKB to 8367 feet TVD. The production casing contains seawater above 8367 feet through the wellhead to the riser. The riser is filled with seawater to 300 feet below the RKB. The top 300 feet is a gel with a density of 16.3 ppg.

This condition would occur if the production casing – reservoir communication occurred after the seawater was introduced into the production casing.

The annulus is filled with mud from the cementing operations. Pressures in the annulus are the same as Condition 1.

The bottom of the production casing is at reservoir pressure. The pressure profile in the production casing is the hydrostatic head of the gel/seawater/mud combination. The hydrostatic pressure of the gel/seawater/mud column is 11083 psi, less than the reservoir pressure of 11844 psi. The fluid level in the production casing will rise as the higher pressure reservoir flows into the casing.

Since the production casing is not sealed at the bottom, lifting the casing will not lift the contents of the casing up to the RKB. Therefore the weight of the contents is excluded from the buoyancy calculation.

The drill string is supported by the rig and has no effect on the buoyancy of the production casing.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or annular or interior fluids was not considered because the value of friction force is dependent on many factors that are unknown. Friction, however, will only apply when the casing moves.

The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1.

The buoyancy calculation sheet (Table 3.5) and results and Figure 3.5 are included in this section.

The production casing and hanger may lift in this condition. When the hanger lifts, the pressures will equalize between annulus and the interior and the fluid in the annulus will flow to the interior. The amount of lift cannot be determined with the available information.



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### Results

- The production casing interior is underbalanced by about 756 psi, so the reservoir will flow into the casing.
- The buoyancy is a negative 0.11 million lb, so the production casing will not lift.
- The net force on the hanger is a positive 0.11 million lb, so the production casing can lift. Note the buoyancy force is included in this force.
- The casing and hanger can lift in this condition. Reservoir fluid has already entered the production casing in this condition, so the effect on flowrate of production fluids is small.
- The amount of lift was not determined.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 9078 psi
  - Interior pressure 6818 psi
  - Vertical force 140896 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 11835 psi
  - Interior pressure 11835 psi
  - Vertical force 118350 lb, upwards

### Condition 6

#### Basis

Communication through production casing cement allows reservoir fluid into both the annulus and the production casing. The production casing contains production fluid in a static column (a flowing well is outside the study scope due to the uncertainty of the flowrate). The annulus also contains a column of static production fluid.

This condition would occur if reservoir communication occurred and if the communication were established between both the annulus and the production casing.

Production fluid properties were determined as in Condition 3. The bottom of the annulus and the production string are at reservoir pressure.

The pressure profile in the production casing is the combination of hydrostatic head of the production fluid and the friction drop of the flowing fluid. The production fluid has either swept the previous gel/seawater/mud fluid out of the production casing or has flowed around it.

The rate of flow is unknown. Since the pressure in the production casing is used only to determine the downward force on the hanger, intermediate pressures and temperatures are not determined.

Since the production casing is not sealed at the bottom, lifting the casing will not lift the contents of the casing up to the RKB. Therefore the weight of the contents is excluded from the buoyancy calculation.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.



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The casing shoe was treated as 7" casing.

Friction between the production casing and cement or annular or interior fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1

The buoyancy calculation sheet (Table 3.6) and results and Figure 3.6 are included in this section.

### Results

- The buoyancy is a negative 0.47 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 0.64 lb, so the hanger will not lift.
- The resultant of the pressure forces on the hanger is a downward force. This is expected because the two pressures are equal and the interior area is larger than the annulus area.
- The bottom of the production casing is assumed to be open so the weight of the contents is not included in the buoyancy calculation.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 10540 psi
  - Interior pressure 10540 psi
  - Vertical force 85346 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 11835 psi
  - Interior pressure 11835 psi
  - Vertical force 118350 lb, upwards

### Condition 7

#### Basis

Communication through production casing cement allows reservoir fluid into the production casing. The production casing contains production fluid in a static column (a flowing well is outside the study scope due to the uncertainty of the flowrate). The annulus contains a mud column from the original cement process.

This condition would occur if the reservoir communication occurred and if communication were established between the production casing but with no communication with the annulus.

Production fluid properties were determined as in Condition 3. The bottom of the production casing is at reservoir pressure. The pressures of the annulus are as in Condition 1.

The pressure profile in the production casing is the combination of hydrostatic head of the production fluid and the friction drop of the flowing fluid. The production fluid has either swept the previous gel/seawater/mud fluid out of the production casing or has flowed around it.

The rate of flow is unknown. Since the pressure in the production casing is used only to determine the downward force on the hanger, intermediate pressures and temperatures were not determined.



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Since the production casing is not sealed at the bottom, lifting the casing will not lift the contents of the casing up to the RKB. Therefore the weight of the contents is excluded in the buoyancy calculation.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or annular or interior fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1.

The buoyancy calculation sheet (Table 3.7) and results and Figure 3.7 are included in this section.

### Results

- The buoyancy is a negative 0.11 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 1.28 million lb, so the hanger will not lift.
- The resultant of the pressure forces on the hanger is a downward force. This is because the interior pressure is higher (8816 psig) than the annulus pressure (3797 psig).
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 9077 psi
  - Interior pressure 10540 psi
  - Vertical force 29872 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 11835 psi
  - Interior pressure 11835 psi
  - Vertical force 118350 lb, upwards

## Condition 8

### Basis

Communication through production casing cement allows reservoir fluid into the production casing. The production casing contains production fluid in a static column (a flowing well is out of the study scope due to the uncertainty of the flowrate). The annulus contains a column of production fluid that is static.

This condition would occur if the reservoir communication occurred and if communication were established between the production casing but with no communication with the annulus.



## Section 3 – Calculations & Figures

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Production fluid properties for the annulus were determined as in Condition 3. The bottom of the production casing is at reservoir pressure. The pressures of the annulus are as in Condition 1.

The pressure profile in the production casing was set at 2358 psi at the TO casing and 11835 psi (reservoir pressure) at the bottom. No attempt was made to model the flowing well fluid and the pressure profile in the casing. The production fluid has either swept the previous gel/seawater/mud fluid out of the production casing or has flowed around it.

The rate of flow is unknown. Since the pressure in the production casing is used only to determine the downward force on the hanger, intermediate pressures and temperatures are not determined.

Since the production casing is not sealed at the bottom, lifting the casing will not lift the contents of the casing up to the RKB. Therefore the weight of the contents is excluded in the buoyancy calculation.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or annular or interior fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1.

The buoyancy calculation sheet (Table 3.8) and results and Figure 3.8 are included in this section.

The hanger will lift in this condition. When the hanger lifts, the pressures will equalize between annulus and the interior and the fluid in the annulus will flow to the interior. We could not determine the amount of lift with the current information.

### Results

- The buoyancy is a negative 0.47 million lb, so the production casing will not lift.
- The net force on the hanger is a positive 0.75 million lb, so the hanger can lift.
- The resultant of the pressure forces on the hanger is an upward force. This is because the interior pressure is lower (2358 psig) than the annulus pressure (8816 psig).
- The pressures and forces at the crossover due to the pressures and area differences were not determined.
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 11835 psi
  - Interior pressure 11835 psi
  - Vertical force 118350 lb, upwards



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## Section 3 – Calculations & Figures

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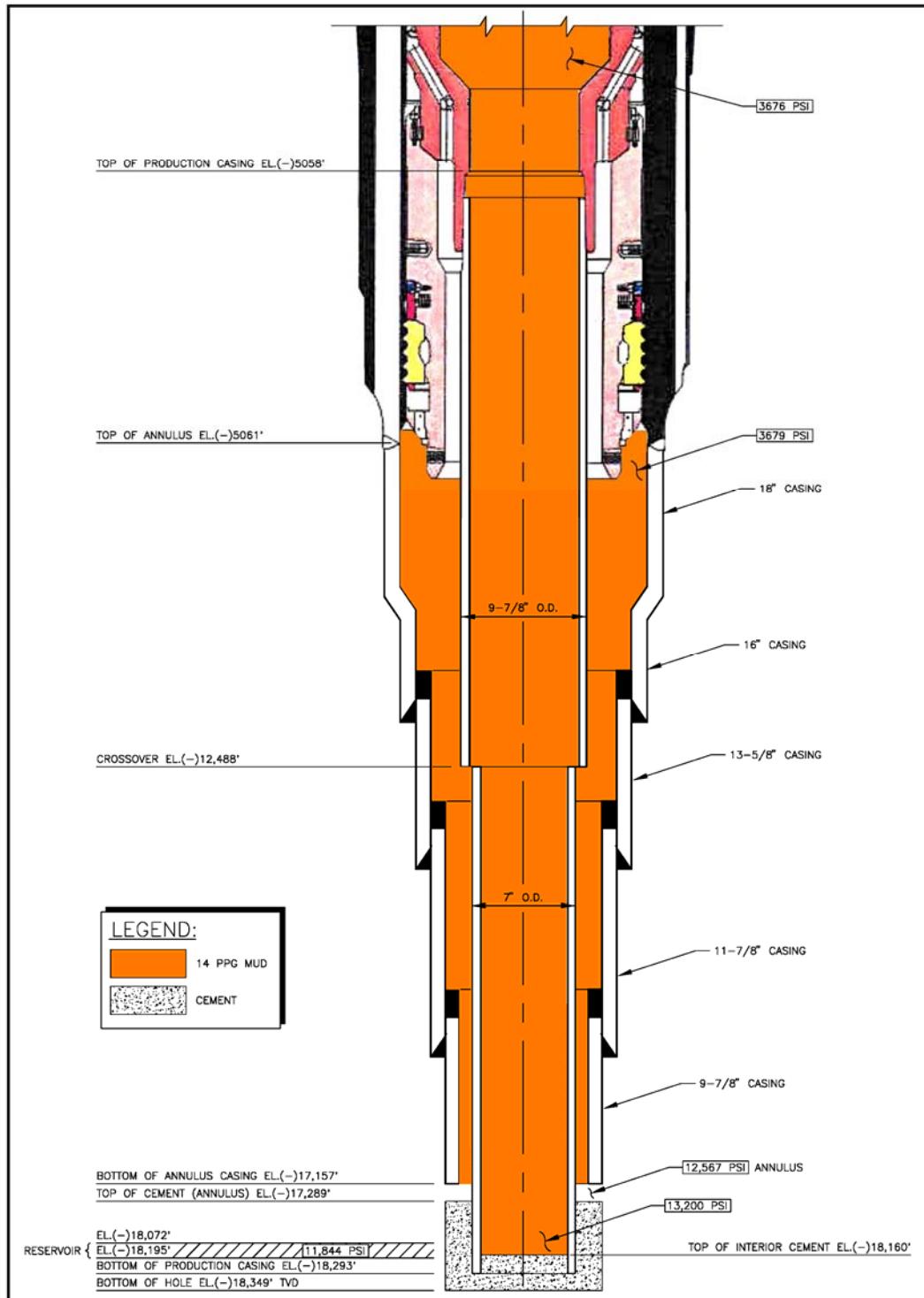


Figure 3.1 – Condition 1

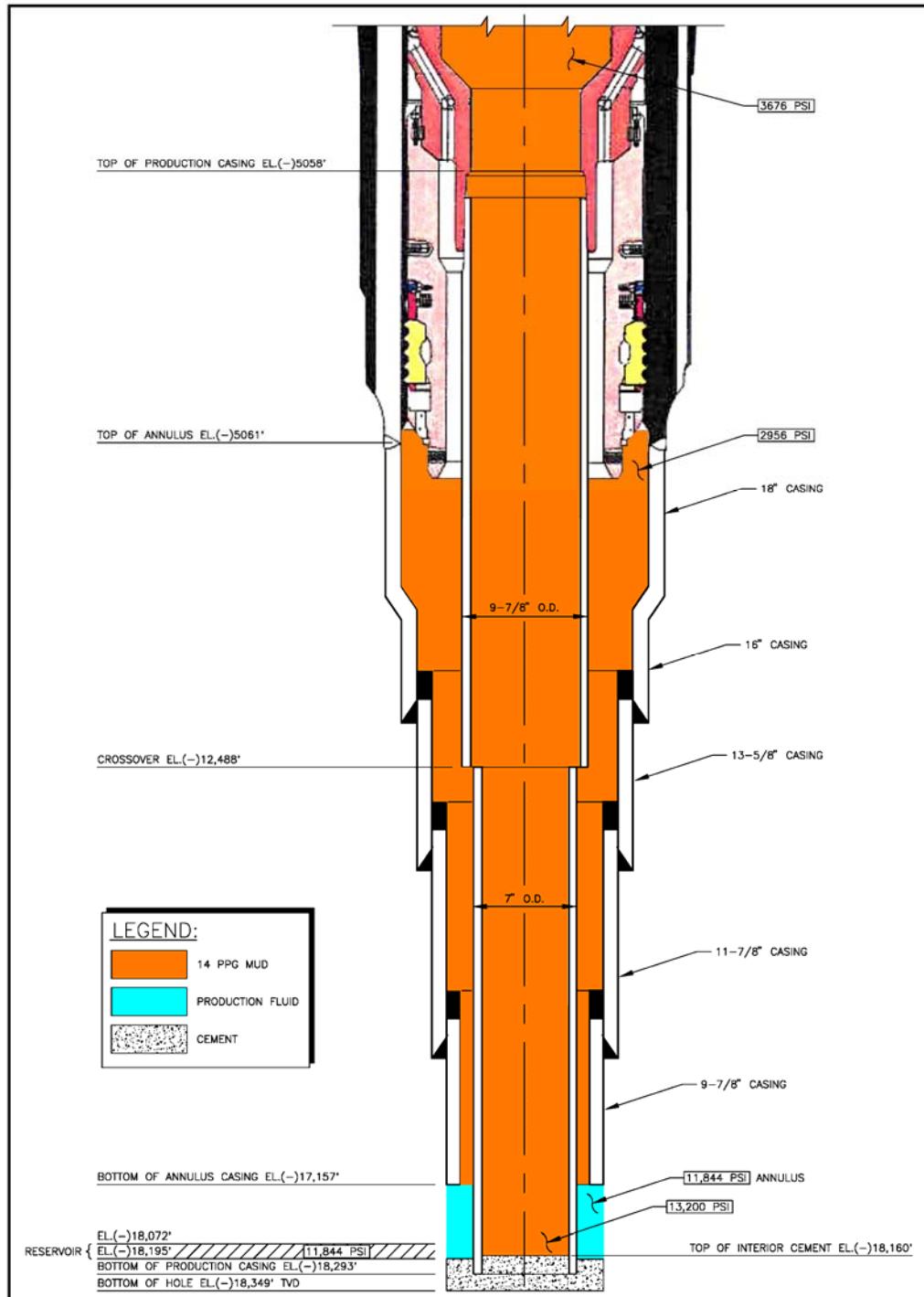


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**Figure 3.2 – Condition 2**



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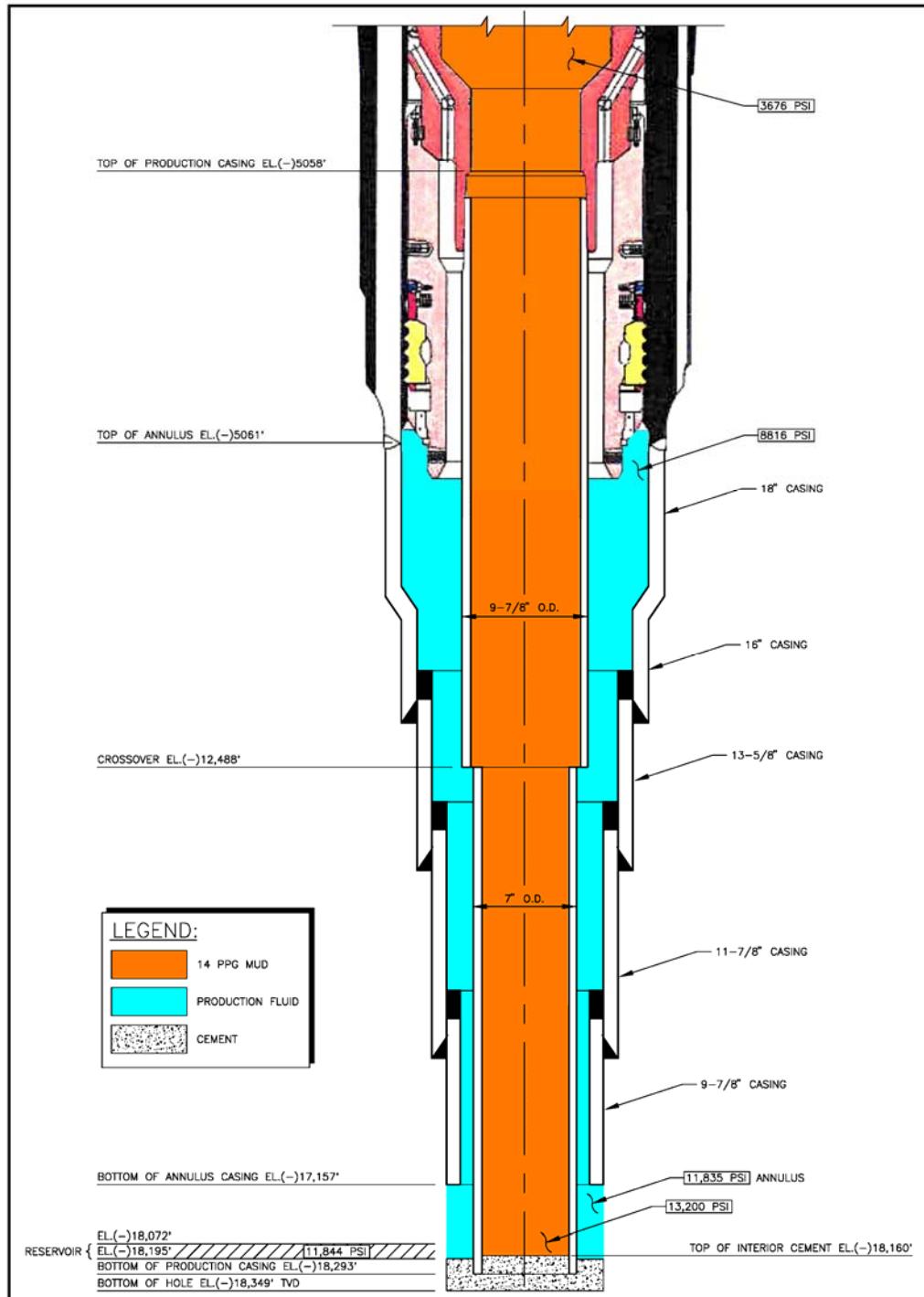


Figure 3.3 – Condition 3



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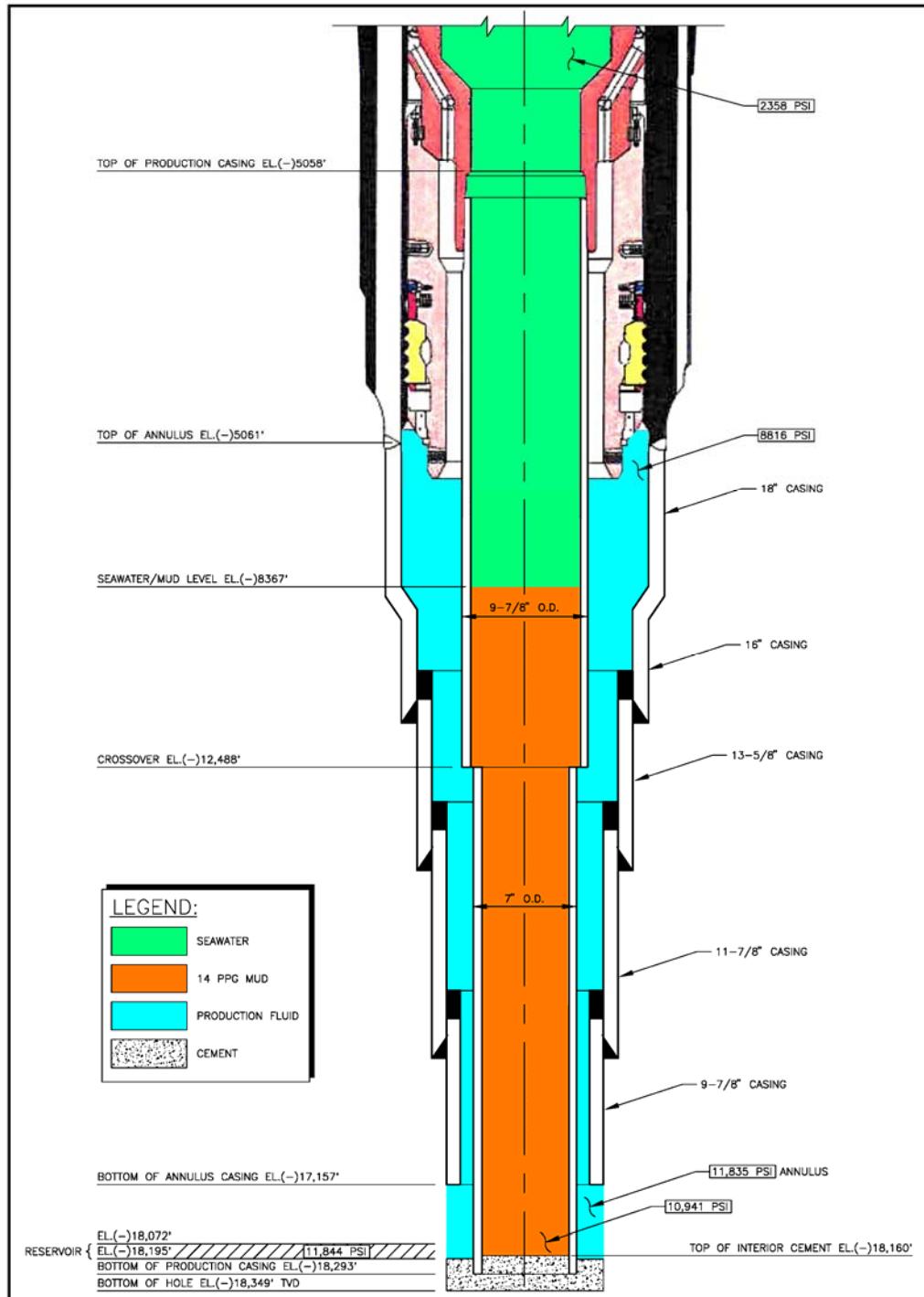
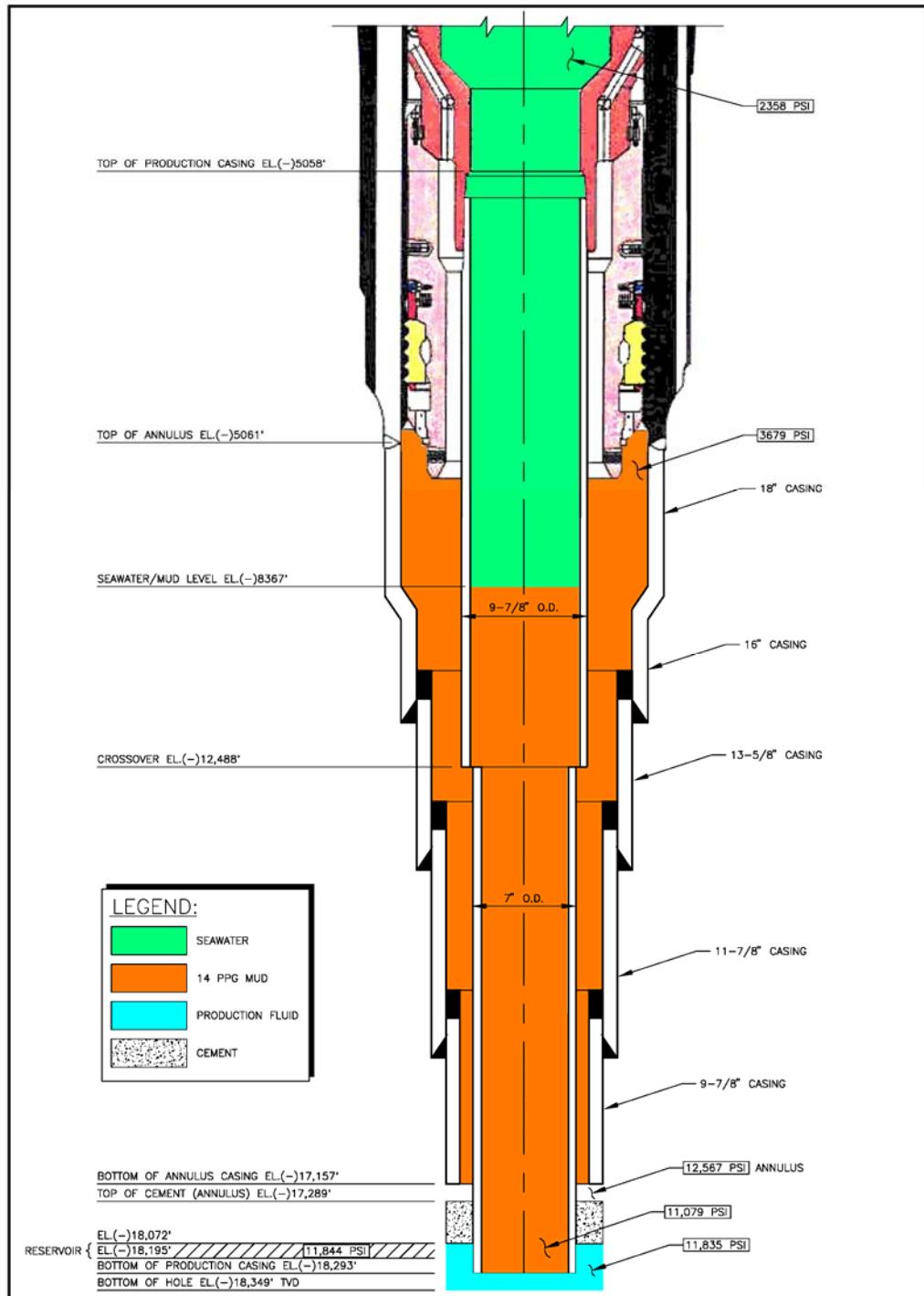


Figure 3.4 – Condition 4

**Section 3 – Calculations & Figures**

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**Figure 3.5 – Condition 5**



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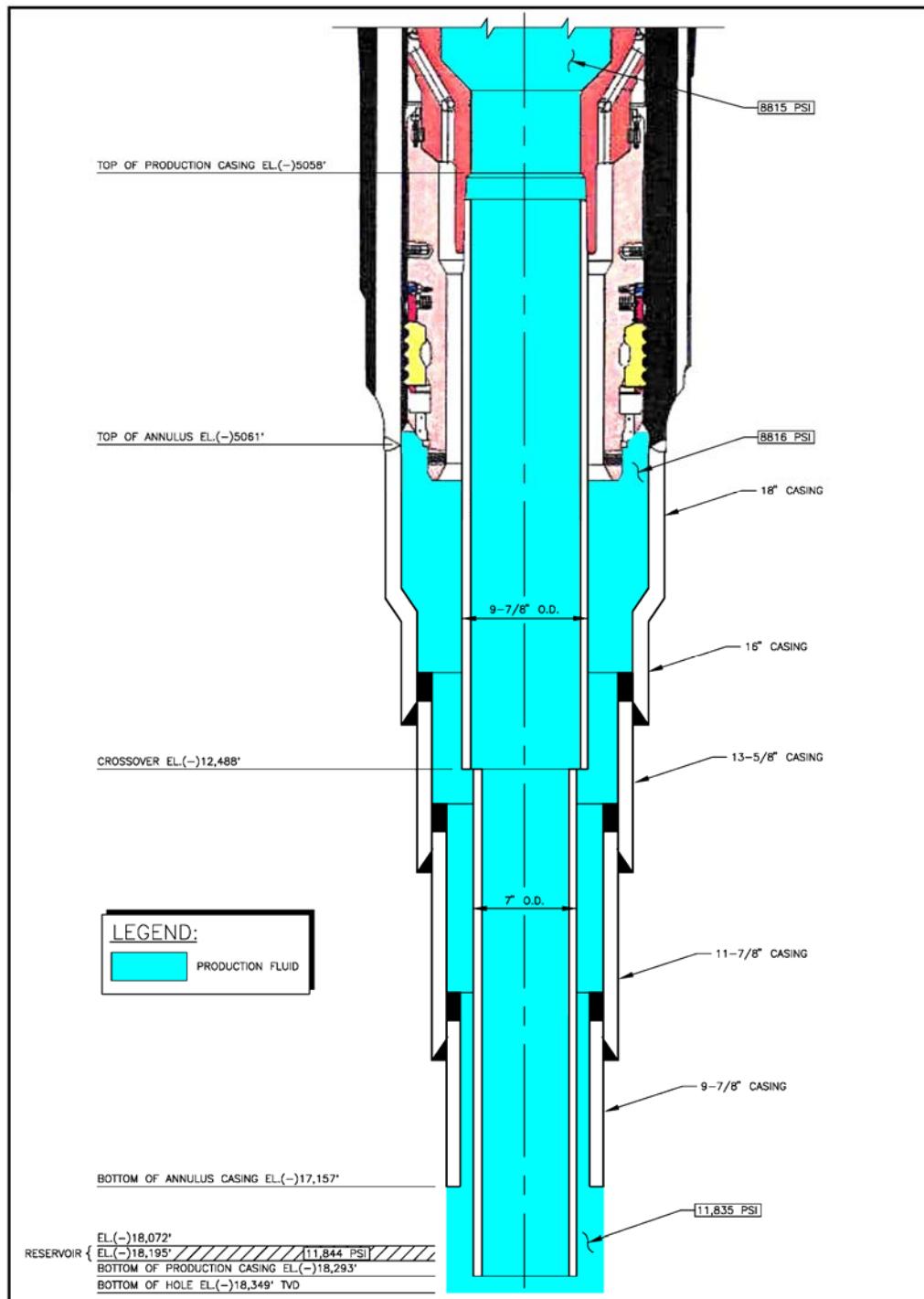
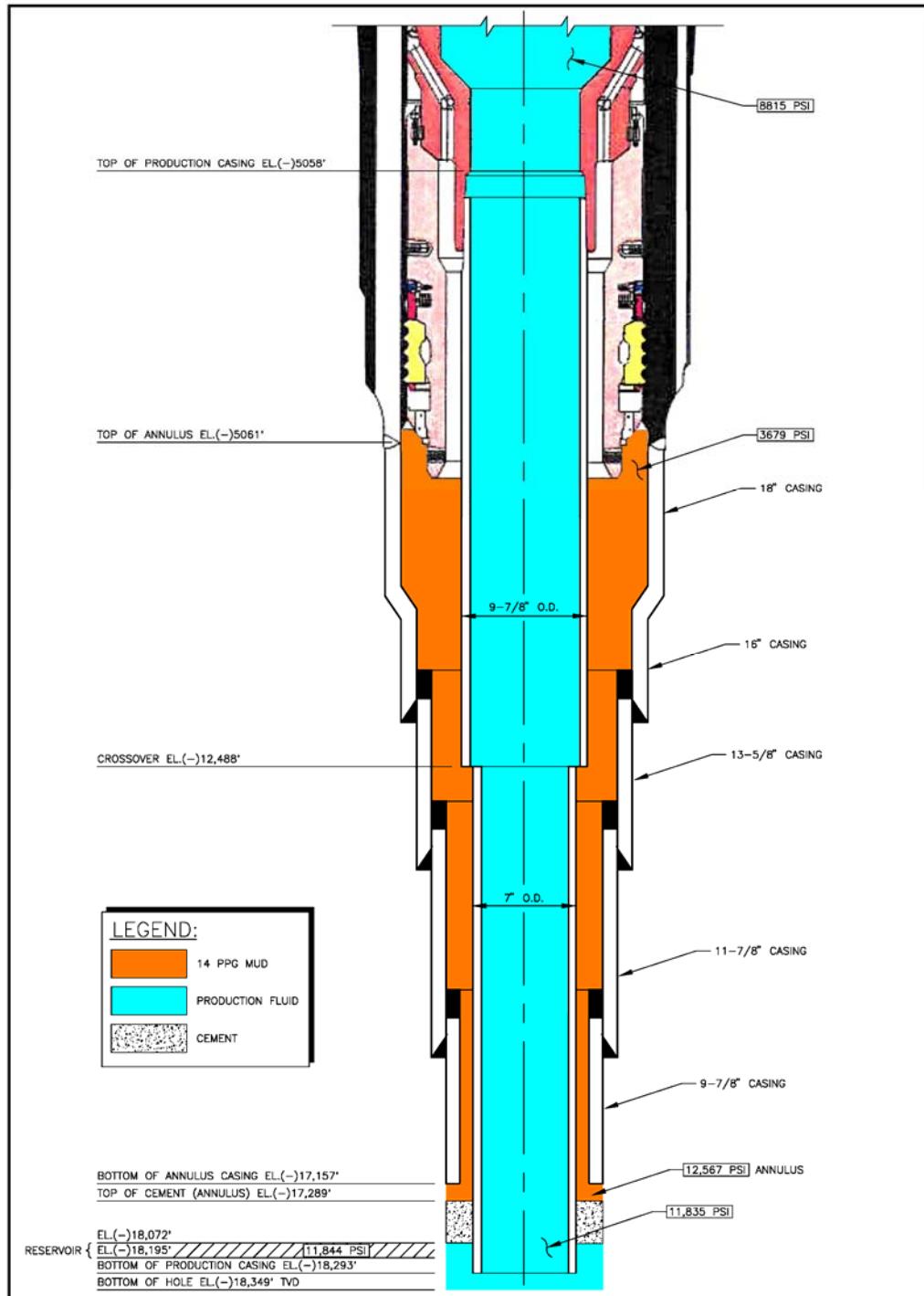


Figure 3.6 – Condition 6

**Section 3 – Calculations & Figures**

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**Figure 3.7 – Condition 7**



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## Section 3 – Calculations & Figures

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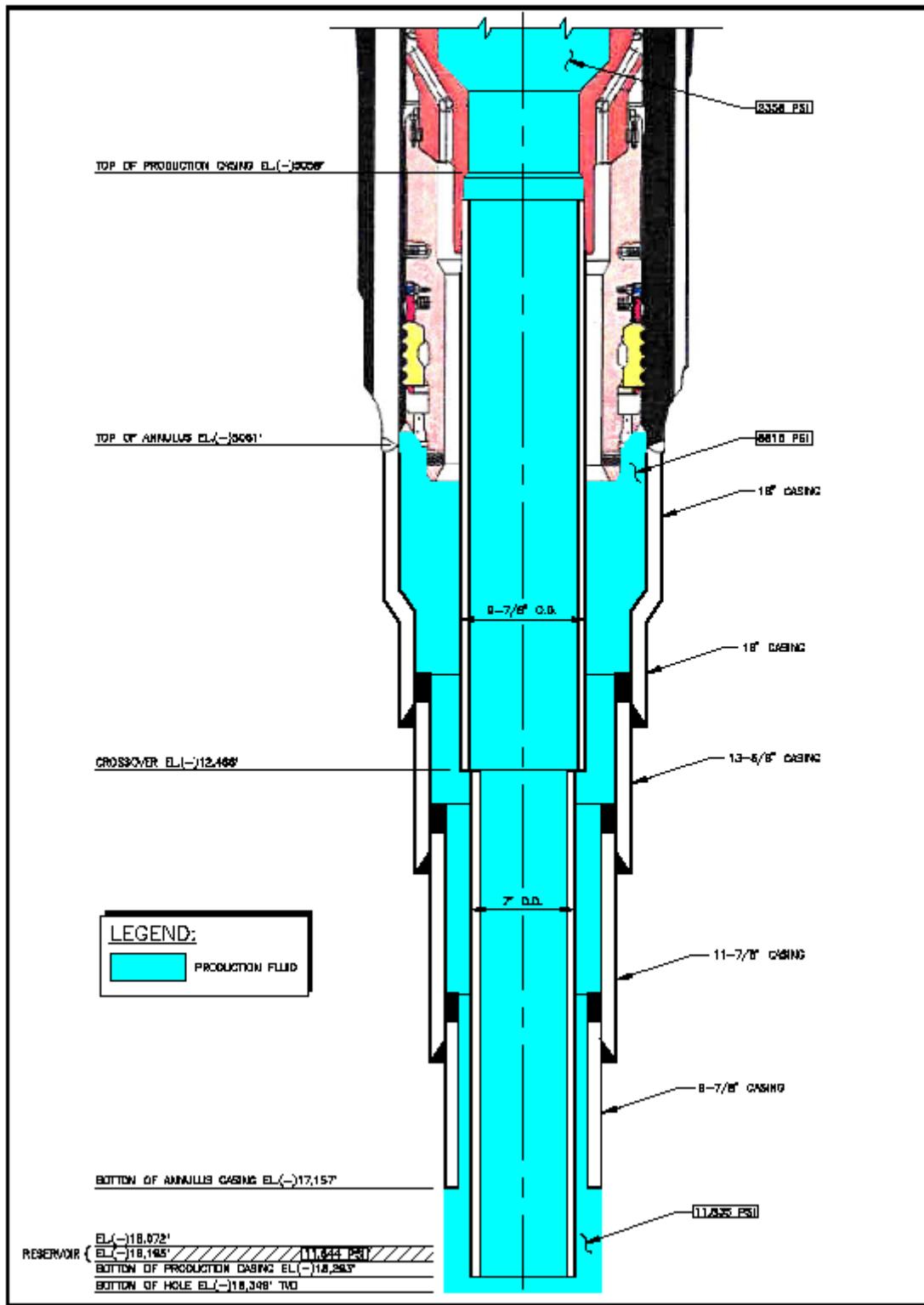


Figure 3.8 – Condition 8





## **Section 3 – Calculations & Figures**

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Notes:

Comment about slugs converts production fluid from flowing into Production Gearing

Windfall Income and Consumption Cycles

Mud fills Production Casing

Since Production Casing is plugged, the contents of the Production Casing affect

Pressure at top of Annulus set equal to balance reservoir pressure at bottom.

Area of Production Casing seal = 195.71 sq

Pressure at top of Annulus = 2956 psi

378434 16

24340 sq miles = 627645 sq km

78998711  
2011.06.25 -

Net Buoyancy (b<sub>m</sub>) = Displaced weight - casting weight + upwards force - downward

+Weight of fluid in ProductionCasing =

Net Buoyancy, lb -1.84E+06

**Tabl2 3.2 – Condition 2**



## **Section 3 – Calculations & Figures**

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**Table 3.3 – Condition 3**

Buoyancy Casing Analysis, Rev. 0

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## **Section 3 – Calculations & Figures**

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Commun

Production fluid fills annulus, is trapped by seals at T O Annulus, no flow of production fluid

Pressure at T O annulus is set by reservoir pressure and density of production fluid.

Cement plug prevents production fluid from flowing into Production Casing.

Cement does not fix Production Casing.

Since Production Casing is plugged, the contents of the Production Casing affect the buoyancy.

Area of Production Casing seal = 195.71 sq in

Pressure at top of Annulus = 8816 psi

Upwards force = 1725401 lb

$$\text{Area of } 18^\circ \text{ seal} = 214.85 \text{ sq in}$$

Pressure wellhead = 2358 psi

Downwards force = 506687 lb

Net Buoyancy,  $b_m$  = Displaced weight - casing weight+upwards force - downwards force

+Weight of fluid in ProductionCasing =

Net Buoyancy, lb -3.39E+05

**Table 3.4 – Condition 4**







### **Section 3 – Calculations & Figures**

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**Table 3.7 – Condition 7**

Buoyancy Casing Analysis, Rev. 0

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## **Section 3 – Calculations & Figures**

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๑๖๔

Communication between reconvict and Production Casing

communications between reservoir and production casing.

Production of fluid in annulus, is trapped by seals at | O Annulus, no flow of production fluid

Pressure at T O annulus is set by reservoir pressure and density of production fluid.

Production fluid is in annulus, cement does not hold Production Casing.

since hole is open, the contents of the Production Casing do not affect the buoyancy.

Area of Production Casing seal =

Pressure at top of Annulus = 8816 psi

1725401 lb  
downwards force =

$$\text{Area of } 18^\circ \text{ seal} = 214.85 \text{ sq in}$$

? pressure wellhead =

— Economic World — 506602 — 2000 — 11

http://www.w3.org/1999/02/22-rdf-syntax-ns#

Weight = Displaced Weight - casting weight + up-ward force - 88 m/hards force

Weight of fluid in Production Casing =

7.46E+05  
Net Buoyancy, lb

**Table 3.8 – Condition 8**



## Section 4 - Basis of Calculations

9/22/2010 Rev. 0

### Temperature Calculations

For temperature growth

$$\Delta L = L * \alpha * \Delta T$$

Where:

L = characteristic length

$\alpha$  = thermal coefficient of material, in/in per Deg F

$\Delta T$  = change in Temperature, Deg F

For 9-7/8" casing, close to the crossover:

$$ID, \text{ in.} = 8.625$$

$$b, \text{ in.} = 4.3125$$

$$t, \text{ in.} = 0.625$$

At 12486 feet depth,

$$T, \text{ temperature at depth, Deg F} = 178$$

$$T_0, \text{ temperature at floor, Deg F} = 70$$

$$\Delta T, \text{ Deg F} = 108$$

$$\alpha = 6.26E-06$$

$$b, \text{ inner id, in.} = 4.3125$$

$$a, \text{ outer id, in.} = 4.9375$$

$$\Delta a, \text{ in.} = a * \alpha * \Delta T, \text{ in.}$$

$$\Delta a, \text{ in.} = 3.34E-03 \quad \text{or} \quad 0.0676 \text{ percent, negligible}$$

$$\Delta b, \text{ in.} = b * \alpha * \Delta T, \text{ in.}$$

$$\Delta b, \text{ in.} = 2.91E-03 \quad \text{or} \quad 0.0676 \text{ percent, negligible}$$

The increase in steel volume is accompanied by a reduction in density.

For 7" casing, close to the shoe:

$$OD, \text{ in.} = 7$$

$$b, \text{ in.} = 3.047$$

$$t, \text{ in.} = 0.453 \quad R/t = 6.726269$$

At 18349 TVD feet depth,

$$T, \text{ temperature at depth, Deg F} = 262$$

$$T_0, \text{ temperature at floor, Deg F} = 70$$

$$\Delta T, \text{ Deg F} = 192$$

$$\alpha = 6.52E-06$$

$$b, \text{ inner id, in.} = 3.047$$

$$a, \text{ outer id, in.} = 3.5$$

$$\Delta a, \text{ in.} = a * \alpha * \Delta T, \text{ in.}$$

$$\Delta a, \text{ in.} = 4.38E-03 \quad \text{or} \quad 0.1253 \text{ percent, negligible}$$

$$\Delta b, \text{ in.} = b * \alpha * \Delta T, \text{ in.}$$

$$\Delta b, \text{ in.} = 3.82E-03 \quad \text{or} \quad 0.1253 \text{ percent, negligible}$$

The increase in steel volume is accompanied by a reduction in density.



## Section 4 – Basis of Calculations

9/22/2010 Rev. 0

### Pressure Calculations

per Roark & Young, "Formulas for Stress and Strain, Fifth Edition:

Table 29, "Formulas for membrane stresses and deformations in thin walled pressure vessels ", pp 448

for vessel with radius R, thickness t under uniform load q (psi)

applicable for  $R/t > 10$

$$\Delta R = q * R^2 / (E * t)$$

E = modulus of elasticity, psi

For 9-7/8" casing, close to the crossover:

$$I D, \text{ in.} = 8.625$$

$$R, \text{ in.} = 4.3125$$

$$t, \text{ in.} = 0.625 \quad R/t = 6.90$$

For 7" casing, close to the shoe:

$$O D, \text{ in.} = 7$$

$$R, \text{ in.} = 3.047$$

$$t, \text{ in.} = 0.453 \quad R/t = 6.73$$

Do not use, R/t is too small.

Table 39, "Formulas for thick walled vessels under internal and external loading", pp 505

For 9-7/8" casing, the maximum differential is at the hanger in condition 4

$$I D, \text{ in.} = 8.625$$

$$R, \text{ in.} = 4.3125$$

$$t, \text{ in.} = 0.625 \quad R/t = 6.9$$

At 5061 foot depth, 48 Deg F

$$\text{inner pressure, psi,} = 2358 \text{ (Condition 4)}$$

$$\text{Outer pressure , psi,} = 8816$$

$$\text{net inner pressure, psi,} = -6458 \text{ Condition 2}$$

$$E, \text{ psi,} = 2.99E+07 \quad \text{Temp} = \text{at 70 Deg F}$$

$$\mu, \text{ Poisson's ratio} = 0.3$$

$$b, \text{ inner id, in.} = 4.3125 \quad L, \text{ length, in.} = 89160$$

$$a, \text{ outer id, in.} = 4.9375$$

$$\Delta a, \text{ in.} = q/E^2*a^2*b^2/(a^2-b^2)$$

$$\Delta a, \text{ in.} = -0.008986 \quad \text{or} \quad -0.182 \text{ percent, negligible}$$

$$\Delta b, \text{ in.} = q*b/E*((a^2+b^2)/(a^2-b^2)+\mu) \quad -0.007197 \quad \text{or} \quad -0.167 \text{ percent, negligible}$$

$$\Delta L, \text{ in.} = -q*\mu*L/E^2*b^2/(a^2-b^2) \quad 37.134928 \quad \text{or} \quad 0.042 \text{ percent, negligible}$$

**Section 4 – Basis of Calculations**

9/22/2010 Rev. 0

**Pressure Calculations (Cont'd)**

For 7" casing, the maximum differential is at the bottom hole in condition 5

O D, in, = 7

R, in, = 3.047

t, in, = 0.453 R/t = 6.726269

At 17163 feet depth,

inner pressure, psi, = 11079

Outer pressure , psi, = 12576

net inner pressure, psi, = -1497

E, psi, = 2.92E+07 Temp = at 262 Deg F

mu, Poisson's ratio = 0.3

b, inner id, in, = 3.047 L, length, in = 68064

a, outer id, in, = 3.50

Delta a, in, =  $q^*/E^2*a^*b^2/(a^2-b^2)$ 

Delta a, in, = -0.001483 or -0.042 percent, negligible

Delta b, in, =  $q^*b/E*((a^2+b^2)/(a^2-b^2)+mu)$ 

Delta b, in, = -0.001181 or -0.039 percent, negligible

Delta L, in, =  $-q^*mu*L/E^2*b^2/(a^2-b^2)$ 

27.59 or 0.041 percent, negligible



**Section 4 – Basis of Calculations**

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**Differential Pressure**

		Casing size, in	Differential, psi	
Condition 1		9-7/8"	7"	
	hanger	Pi, psig	3676	
		Po, psig	3676	0
	xovr	Pi, psig	9077	
		Po, psig	9077	0
	BH	Pi, psig		12567
		Po, psig		13200
				-633
Condition 2		9-7/8"	7"	
	hanger	Pi, psig	3676	
		Po, psig	2956	-720
	xovr	Pi, psig	9077	
		Po, psig	8358	719
	BH	Pi, psig		13200
		Po, psig		11844
				1356
Condition 3		9-7/8"	7"	
	hanger	Pi, psig	3676	
		Po, psig	8816	-5140
	xovr	Pi, psig	9077	
		Po, psig	10540	-1463
	BH	Pi, psig		13200
		Po, psig		11844
				1356
Condition 4		9-7/8"	7"	
	hanger	Pi, psig	2358	
		Po, psig	8816	-6458
	xovr	Pi, psig	6818	
		Po, psig	10540	-3722
	BH	Pi, psig		10941
		Po, psig		11835
				-894
Condition 5		9-7/8"	7"	
	hanger	Pi, psig	2358	
		Po, psig	3679	-1321
	xovr	Pi, psig	6818	
		Po, psig	9078	-2260
	BH	Pi, psig		11079
		Po, psig		12567
				-1488
Condition 6		9-7/8"	7"	
	hanger	Pi, psig	8815	
		Po, psig	8816	-1
	xovr	Pi, psig	10540	
		Po, psig	10540	0
	BH	Pi, psig		11835
		Po, psig		11835
				0

**Section 4 – Basis of Calculations**

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**Differential Pressure (Cont'd)**

		Casing size, in	Differential, psi
Condition		9-7/8"	7"
Condition 7	hanger	Pi, psig	8815
		Po, psig	3679 5136
	xovr	Pi, psig	10540
		Po, psig	9077 -1463
Condition 8	BH	Pi, psig	11835
		Po, psig	11835 0
		9-7/8"	7"
	hanger	Pi, psig	2358
		Po, psig	8816 -6458
	xovr	Pi, psig	NA
		Po, psig	NA
	BH	Pi, psig	11835
		Po, psig	11835 0

**Material Properties**

Low Chrome Moly steels, Cr less than 3%

Modulus of elasticity from ANSI B31.3

Temp, Deg F E, psi (multiply table by 10^6)

-100	30.4
70	29.9
200	29.5
300	29.0
	178 29.6
262	29.19
162	29.6

Mean Coefficient of thermal expansion between 70 Deg F and temperature, from ANSI B31.3

Temp, Deg F A, in/in (multiply table by 10^-6)

70	0
125	6.19
150	6.25
175	6.31
200	6.43
250	6.49
275	6.54
	178 6.26
262	6.52
162	6.28



## Section 4 – Basis of Calculations

9/22/2010 Rev. 0

### Volumes and weights of Displaced Fluids

19.5" ID conductor =	.3694 bbl/ft =	15.419 gal/ft
18.375 " ID casing =	.3280 bbl/ft =	13.776 gal/ft
14.92" ID casing =	.2162 bbl/ft =	9.080 gal/ft
9-7/8" OD casing =	.09473 bbl/ft displacement =	3.979 gal/ft
9-7/8" OD, 8.626 " ID casing =	.07227 bbl/ft =	3.035 gal/ft
7" OD casing, displacement =	.0476 bbl/ft =	1.999 gal/ft
7" OD, 6" ID casing =	0.036071 bbl/ft =	1.515 gal/ft

9-7/8" OD, 62.8 #/ft casing is 8.626 "

7" OD, 32 #/ft casing is 6" ID

16", 97#/ft casing is 14.92" ID

"Prod. Casing Fluid ppg" used density of seawater at 80 Deg F

Fresh water = 8.318 lb/gal at 80 Deg F (Ref. No. 9)

Seawater sp gr = 1.025

Seawater = 8.526 lb/gal

Production Fluid density is determined from HYSYS runs which give static gradient at each depth.

gradient (in/ft) / 12 = gradient (psi/ft) psi/ft \* 8.337 / .433 = lb/gal.

7.38257 inH2O/ft= 0.26475 psi/ft = 5.098 lb/gal

Density of seawater at depths is read from the graph of density vs. sea depth, (Ref. No.2)

This data is generic, site specific data was not used.

Density of mud was fixed at 14 ppg. No data was given for the change in density with pressure or temperature.

"Prod Casing Pressure at end of segment" is determined from:

Fluid ppg / 8.37 (freshwater ppg) \* .433 (gradient of freshwater) \* length of segment plus previous pressure

"Prod Casing Fluid Capacity" is determined from:

Either id of casing or tables (Ref. No. 2)

Temperatures of fluid and casing were determined by:

For pumped seawater, temperature is 80 Deg F.

For mud in Production Casing, temperature is in equilibrium with the formations.

Temperatures of the formations were determined from Ref. No. 1

Temperatures of produced fluids were determined from Appendix B, Item 6.

Reservoir pressure was determined from Halliburton, Appendix B Item 5.

Annulus Casing Description is not applicable for the segments of the riser, BOP and the Production Casing hanger.

These items are supported by the well head and surface conductor and do not affect the loads on the Production Casing. The contents of the riser, BOP and Production Casing hanger would move if the Production Casing moves.

Therefore, the contents of the riser, BOP and Production Casing hanger are included in the Production Casing loads.



## Section 4 – Basis of Calculations

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### Crossover Dimensions

Crossover dimensions from K&B drawing. (See Appendix B, Item.4)

1 Cross over is 10.078" od, 8.626" id for 8.5"

2 Cross over is 10.078" od, 6.034" id at 48-32.161-8.5 = 7.34"

1.5 Tapers from 1 to 2 over the 7.34"

3 Cross over is 10.078" od, 6.034" id for 32.382-48+32.151"=16.533"

4 Cross over is 7.016" od, 6.034" id at 6.9"

3.5 Tapers from 3 to 4 over 48-32.392-6.9 = 8.708"

so:

Taper is  
10.087 " od by 8.626 " id for 8.5 " , volume of OD = 679.26 cu in  
" , volume of ID = 496.74 cu in 0.2875 cu ft = 2.15 gal

10.087 " od by 7.33 " id for 7.34 " , volume of OD = 586.56 cu in  
" , volume of ID = 309.74 cu in 0.9990 cu ft = 7.47 gal

10.087 " od by 6.034 " id for 16.533 " , volume of OD = 1321.19 cu in  
" , volume of ID = 472.77 cu in

Taper is

8.5515 " od by 6.034 " id for 8.708 " , volume of OD = 500.14 cu in  
" , volume of ID = 249.01 cu in  
7.016 " od by 6.034 " id for 6.9 " , volume of OD = 266.76 cu in  
" , volume of ID = 197.31 cu in

Displacement is total of OD volume = 3353.90 cu in = 1.94 cu ft = 14.51 gal  
Contents is total of ID volume = 1725.57 cu in = 0.999 cu ft = 7.47 gal  
Weight is difference of OD volume and ID volume = 1628.33 cu in = 470.59 lb  
Density is 8000 kg/cu m = 0.289 lb/cu in

Check totals

Add section 1.4

47.981 Overall length from drawing = 48 inches  
Length is ok.



## Section 4 – Basis of Calculations

9/22/2010 Rev. 0

### Seal Area Calculations

Pressure, force downward

Pressure = pressure in wellhead = pressure at top of Production casing = pressure at 5058 foot depth

Area = inside of 18.620" Seal = Area (18.62" dia) - Area (8.553" dia)

214.8461 sq in

Pressure, force upwards

Pressure = pressure in annulus at T O Annulus = pressure at 5061 foot depth

Area = inside of 18.620" Seal = Area (18.62" dia) - Area (9.875" dia)

195.7124

### Forces at Crossover

Crossover dimensions from K&B drawing, (See Appendix B, Item.4)

1 Cross over is 10.078 " od, 8.626" id at upper end

4 Cross over is 7.016" od, 6.034" id at lower end

External area

The external vertical force on the crossover is the external pressure times the horizontal projection of the outside area

The internal vertical force on the crossover is the internal pressure times the horizontal projection of the inside area

Outside area, below reducer = 10.087 OD by 7.016" ID = Ao = 54.14 sq in

Outside area, above reducer = 10.087 OD by 9.875" ID = Ao = 16.21 sq in

Inside area = 8.625" OD by 6.034" ID = Ai = 29.83 sq in

Note the two outside area counteract each other.

Pressure forces

Pressure forces are due to the different pressures and areas at the crossover.

Radial forces.

The Horizontal forces are due to the difference in the external and internal pressures.

The pressures are listed below because the radial pressures are used to determine stress levels in the casing.

Condition	External Pressure, psi	Internal Pressure, psi
1	9077	9077
2	8354	9077
3	10540	9077
4	10540	6818
5	9078	6818
6	10540	10540
7	9077	10540



## Section 4 – Basis of Calculations

9/22/2010 Rev. 0

### Forces at Crossover (Cont'd)

#### Vertical Forces

The external vertical force on the crossover is the external pressure times the horizontal projection of (the below outside area - the above outside area.)

The internal vertical force on the crossover is the internal pressure times the horizontal projection of the inside area

The forces oppose each other because the external vertical force is an upward force, and the internal force is an downwards force.

For simplicity, the same external pressure is used for the above and below outside areas.

These areas are only 4 feet apart.

Condition	Net		
	External Pressure, psi	Internal Pressure, psi	upwards vertical force, lbm
1	9077	9077	73500
2	8354	9077	46078
3	10540	9077	128988
4	10540	6818	196363
5	9078	6818	140896
6	10540	10540	85346
7	9077	10540	29872

These forces are included in the buoyancy calculations by Archimedes principle.



## Section 5 - HYSYS Calculations

9/22/2010 Rev. 0

### WELL SIMULATION USING ASPEN HYSYS (ver. 7.1)

The well bore sample report from Schlumberger (Appendix B) was reviewed for the best fit for the component input requirements for HYSYS. The sample chosen was sample 1.18. The component information supplied was entered into the computer for simulation. Components heavier than C30 were combined to form a C30+ component. Since the oil/gas composition is eighty-five mole percent gas, the C30+ component will not significantly change the simulation.

Once the input stream compositions were entered and a physical property database chosen, the process flow for the well simulation was modeled in HYSYS. The sample component results were entered for the gas and the oil phases at a set reservoir pressure of 11,856 psia and reservoir temperature of 243 degrees Fahrenheit. A mixer was used to combine the phases into one stream at a 11,856 psia and 243 degrees Fahrenheit, then sent through a heat exchanger to adjust temperature and pressure to the sample collection conditions of 236 degrees Fahrenheit and 11,835.3 psig. The output was connected to a pipe with a length and vertical elevation specified to match the bottom hole to RKB TVD. The exit temperature was also specified. The velocity in the pipe was minimized in order to approximate a static column of fluid. The chosen output numbers were density, temperature and pressure at various elevations. These outputs were inputs into the buoyancy calculation worksheets.

The specified output temperature was chosen to be 110 deg F at rig floor elevation. This approximates a normal wellhead temperature for this type of production.

In this simulation, the pipe length and elevation (well inlet is zero length at zero elevation) were set to 18,074 feet. The molar flow was set to 100 moles per hour to make checking of the component composition percentages easier and to minimize the flow through the pipe (i.e. Well) to approach static conditions.

The well profile output was extracted and loaded into an Excel spreadsheet. The density and pressure were used in the buoyancy spreadsheets.

To check sensitivity to temperature, the specified output temperature was changed to 160 Deg F at rig floor elevation. All other parameters were held constant. The output for the 110 Deg F outlet temperature case was compared to that for the 160 outlet temperature case. The pressure differences at the production casing hanger were different by only 0.36%. For the buoyancy calculations, this difference is negligible, so the pressures are insensitive to the chosen outlet temperature.

The temperature of the static column will drop with time due to lower temperatures in the formations eventually reaching the formation temperatures. This process will take an undetermined length of time. The densities of the fluid will be higher so the hydrostatic pressures will be lower at the top of the column. (The bottom pressure is controlled by the reservoir pressure.) The higher temperatures used in the model therefore result in higher pressures at the hanger. The results in the buoyancy calculations are an upper bound for pressures at the hanger.

The buoyant force that lifts the casing in Condition 4 will be lower as the production fluid cools.



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## Section 6 - List of References

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2. Schlumberger\i-Handbook
3. Crane "Flow of Fluids" Technical Paper No. 410 Physical Properties of Water, pg-A6
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## Appendix A – List of Abbreviations

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KEI	Keystone Engineering Inc
BOEMRE	Bureau of Ocean Energy, Management, Regulation and Enforcement
RKB	Rig Kelly Bushing
ppg	pound per gallon
SOBM	Synthetic oil based mud
T O	Top Of
BH	Bottom Hole
BOP	Blow Out Preventer
TVD	True Vertical Depth
sq in	Square Inch
psi	Pounds per square inch
lb	Pounds
wt	Weight
x-over	crossover
Deg F	Degrees Fahrenheit
ID	inner diameter
OD	outer diameter
bbl	barrel (42 gallons)



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## Appendix B – Material Supplied by BOEMRE

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