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BUREAU OF OCEAN ENERGY  
MANAGEMENT, REGULATION, AND ENFORCEMENT

## Buoyancy Casing Analysis

REPORT

Rev. 0

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## Quality Assurance

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The scope of this report is limited to the matters expressly covered. This report is prepared exclusively for and at the request of the BOEMRE. In preparing this report, Keystone Engineering Inc has relied on information provided by the BOEMRE. KEI has made no independent investigation as to the accuracy of the information provided and has assumed the information is accurate and complete.

All findings and conclusions stated in this report are based upon the information as it existed at the time of the report. Changes to the information may affect the findings and conclusions.

In the course of this investigation, KEI had conversations with persons not on the investigation team and this report draws upon the members understanding of those conversations. The investigation team did not record or produce verbatim transcripts of those conversations nor did the team ask the other parties to review or endorse the notes taken by the team members.

Graphics are used to depict information and conditions; these may be simplified or not to scale. They are intended as an aid to the reader in the context of the report.

The view, opinions, and/or findings contained in this report are those of the author(s) and should not be construed as an official Government position, policy or decision, unless so designated by other documentation.



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## Section 1 - Introduction

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### Objective

Keystone Engineering Inc (KEI) was contracted by the Bureau of Ocean Energy Management, Regulation and Enforcement (BOEMRE) to review the design of the BP Macondo Well 9-7/8" by 7" production casing and its hanger and to calculate the static forces acting upon the casing to determine if they could lift the casing and hanger from its design location. The deliverables are:

- Calculation of buoyancy force acting on the 9-7/8" by 7" production casing
- Calculation of forces on the
  - Bottom of the 7" casing
  - Crossover of the 9-7/8" by 7" casing
  - 9-7/8" by 7" casing hanger

The BOEMRE provided KEI with data that included a wellbore schematic, a casing crossover schematic, reports from cement service providers and drilling reports. The data is listed in the Appendix B.

### Method

A spreadsheet model of the wellbore was developed which considered the properties of the annulus casing, production casing, fluids in both the annulus and production casing, and formation temperatures at wellbore depths. This data was derived from the information provided by BOEMRE.

Several possible operating conditions were considered and eight were chosen for detailed calculations. The conditions are:

- Condition 1 - Wellbore as designed with the cement complete as designed. The design includes 14 ppg mud in the annulus up to the seal/hanger at the 5061 foot depth and the production casing up to the Rig Kelly Bushing (RKB). (See Fig 3.1.)
- Condition 2 - Communication through the cement to the annulus that allows the pressure at the bottom of the annulus to drop to the reservoir pressure. The annulus is still full of 14 ppg mud up to the seal at the 5061 feet. Production casing is full of 14 ppg mud. (See Fig 3.2.)
- Condition 3 - Communication through the cement to the annulus that allows production fluid into the annulus. Annulus is filled with production fluid up to the seal at the 5061 feet. Production casing is full of 14 ppg mud. (See Fig 3.3.)
- Condition 4 - Communication through the cement to the annulus that allows production fluid into the annulus. Annulus is filled with production fluid up to the seal at the 5061 feet. The production casing is filled with gel, seawater and mud. The mud level is at 8367 feet, which is the depth of the drill string in the wellbore schematic. (See Fig 3.4.)
- Condition 5 - Communication through the cement that allows production fluid into the production casing. Annulus is filled with 14 ppg mud up to the seal at the 5061 feet. The production casing is filled with gel, seawater and mud. The mud level is at 8367 feet which is the depth of the drill string in the wellbore schematic. (See Fig 3.5.)
- Condition 6 - Communication through the cement that allows production fluid into the production casing. Annulus is filled with production fluid up to the seal at the 5061 feet. The production casing is filled with static production fluid. (See Fig 3.6.)



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- Condition 7 - Communication through the cement that allows production fluid into the production casing. Annulus is filled with 14 ppg mud up to the seal at 5061 feet. The production casing is filled with static production fluid. (See Fig 3.7.)
- Condition 8 - Communication through the cement that allows production fluid into the production casing. Annulus is filled with production fluid up to the seal at 5061 feet. The production casing is filled with static production fluid. The pressure in the casing at 5061 feet is set to 2358 psi, which is the pressure from Condition 5 with gel and seawater in the casing above the wellhead. This is a close approximation to the pressure if the production fluid is flowing to the surface. The production fluid is not modeled in HSYS since the flow rate is not known. This condition will give an estimate of the highest pressure in the annulus and the lowest pressure in the casing. (See Figure 3.8)

These conditions are detailed in Section 3 of this report. Diagrams showing the fluid levels and the pressures are included for each condition.

The production casing was considered to have been set on its hanger while the cement was still fluid so the weight of the casing is supported by the hanger only and no axial support is given by the cement or the bottom of the hole. Without this assumption, the forces on the hanger cannot be determined.

The buoyant forces in each condition are developed using the principle of Archimedes that the buoyant force on a submerged object is the weight of the displaced fluid. The force varies depending on the weight of the fluid in the annulus; in this study the fluid is either 14 ppg mud or production fluid from the reservoir. The shape of the object has no effect on the buoyant force, so the pressure differentials at the 9-7/8" by 7" crossover have no effect.

The differential pressure between the annulus and the interior of the production casing will cause the casing dimensions to change from the nominal design. The amount of deformation was calculated to determine the effect. The temperature of the casing will also change the dimensions, and those changes were determined. The calculations are included in Section 4.

The production casing is made of Q-125 steel (9-7/8") and HC-125 steel (7"). These steels are classified as Low Chrome Moly steels. The properties of Chrome Moly steels were taken from the table for "Low Chrome Moly steels, chrome < 3%" from ANSI B31.3. The properties are included in Section 4.

The dimensions of the 9-7/8" by 7" crossover were taken from the drawing by K&B (Appendix B). The weight and volumes of the crossover were calculated separately and entered into the buoyancy calculations as a combined weight and volume. The details are included in Section 4.

Differences in the annular and interior pressures at the crossover will result in net radial forces at the crossover. Also, due to the shape of the crossover (larger top reducing to a smaller bottom), the areas exposed to the annular and interior pressures differ. The crossover is therefore subject to a net vertical force that is calculated in Section 4. Note that this vertical force is included in the buoyancy force and is not additional. The principle of Archimedes uses displacement weights, the shape of the object (hull) is not considered.



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The temperature and compressibility properties of the Synthetic Oil Based Mud (SOBM) were not available so a constant 14 ppg density was used.

The density of the production fluid was developed by entering the component analysis from Schlumberger (Appendix B, Item 6) into a HYSYS model for a static column from the bottom of the well to the hanger. The boundary conditions for the column are the bottom hole temperature and pressure and an upper temperature of 110 Deg F at the rig floor. The upper temperature was chosen to match wellhead temperatures commonly observed in production operations. The HYSYS model is detailed in Section 5.

The capacity of the casing strings is calculated in Section 4.

The forces on the bottom of the casing were determined from the following condition. The production casing and its annulus were filled with mud and the cement was pumped into the bottom of the well with the casing elevated above the hanger. The casing was then lowered into the cement, which was still fluid. The casing was then supported by the hanger. The pressures on the annulus and interior sides are equal, and the axial force is that pressure times the area of the 7" 32 lb/ft casing. The axial force is then the hydrostatic pressure (13,200 psi) times the casing metal area (10 sq in). Three conditions are possible:

- The cement seals the casing interior but allows communication between the annulus and the reservoir. The axial force remains the same, 132,000 lb.
- The cement allows communication between the interior and the reservoir but seals the annulus. The hydrostatic force is the reservoir pressure.
- The reservoir communicates with both the annulus and the interior. The hydrostatic force is the reservoir pressure.

The resulting forces and pressures are listed in the results for each condition. Note that these forces are included in the buoyant force calculation and are not to be added to the buoyant force.

Temperatures along the casing string will change with fluid flow from the reservoir. The casing was assumed to be in equilibrium with the formation temperatures when the casing is set. Determination of casing temperatures depends on many factors (e.g. fluid properties, flow rates, cement and annular fluid properties, reservoir properties and time) that are not known. The determination of forces due to temperature expansion in the casing is beyond the scope of the investigation.



## Section 2 - Executive Summary

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The design of the 9-7/8" by 7" production casing for the Macondo well was reviewed for determination of the buoyant force that would lift the production casing from its hanger. Eight possible operating conditions of the wellbore were examined. The key results are listed below:

- Conditions 5 and 8 result in a net upward force on the hanger.
- The weight of the production casing steel is greater than the weight of the displaced fluid if the fluid is 14 ppg mud. Since the mud is the heaviest of the displaced fluids, the production casing is not buoyant if the pressure forces at the hanger are not considered.
- The weight of the contents of the production casing opposes the buoyant force if the cement at the bottom of the casing seals the casing. The weight of the contents is the largest force in the calculation.
- The net buoyancy of the production casing is:
  - the weight of the displaced fluid
  - minus the weight of the steel
  - minus the weight of the contents of the casing (if the cement seals the contents)
- If the cement at the bottom of the casing does not seal the casing, the weight of the contents has no influence on the buoyant force.
- The net force on the production casing hanger is:
  - the net buoyancy of the casing
  - plus the annulus pressure at the hanger times the projected annular area of the hanger
  - minus the interior pressure of the casing times the projected inside area of the casing
  - This calculation assumes the production casing is set with no support from the cement or the bottom of the hole, i.e. the casing is set on fluid cement.
- The annulus pressure and pressure inside the production casing at the hanger create large forces.
  - If the cement at the bottom of the casing does not seal, the net buoyancy becomes positive (upwards) if the annulus pressure is greater than the interior pressure by a certain amount. This amount changes for each condition depending upon the density of the fluids in the annulus and interior.
  - If the cement seals, the net buoyancy will be negative (downwards) regardless of annulus pressure.
- The temperatures and pressures of the well do not change the radial dimensions of the production casing enough to be significant. The changes are about 0.13% (temperature of 262 Deg F) and 0.18% (differential pressure of 6572 psi).
- The production fluid static column was modeled in HYSYS using reference temperatures of 110 and 160 Deg F to determine sensitivity to the assumed temperature. The net change was 0.36% which is considered negligible.
- Forces due to temperature changes of the production casing were not determined.





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### Condition 1

#### *Basis*

The wellbore is filled with mud fluid and the cement is intact as designed.

The production casing is filled with mud to the Rig Kelly Bushing (RKB).

The annulus is filled with mud to the production casing hanger.

Cement seals the reservoir from the annulus and the production casing, so the pressures in the well are hydrostatic pressures of the mud.

Since the production casing is sealed at the bottom, lifting the casing will lift the contents of the casing up to the RKB, and so the weight of the contents is included in the buoyancy calculation. This weight of the contents will act to oppose the buoyancy force.

The hydrostatic pressure in the annulus at the bottom of the hanger is due to mud fixed in the annulus after cementing when the production casing is lowered into the hanger. This annulus pressure acts upon the exposed annular area of the production casing hanger creating an upward force that adds the buoyant force. This upward force is opposed by the hydrostatic pressure in the production casing acting on the exposed interior area of the hanger.

The area of the production casing hanger exposed to the annulus pressure is the area between the inner diameter of the 18-3/4 inches casing (18.62 inches) and the outer diameter of the production casing hanger (9-7/8 inches). This area was determined to be 195.7 sq. in. (from the Dril-Quip drawing 2-PD-32424-02CP, Appendix B, Item 3).

The area of the production casing hanger exposed to the production casing pressure is the area between the inner diameter of the 18-3/4" hanger (18.62 inches) and the inner diameter of the production casing hanger (8.553 inches). This area was calculated to be 214.8 sq. in. (from the Dril-Quip drawing 2-PD-32424-02CP, Appendix B, Item 3).

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or (annular or interior) fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the cement to hold the production casing in place was not considered.

The shoe and bottom of the hole formation will resist the lowering of the production casing. Since this effect is applicable only if the casing is lowered by a negative buoyant force and this study is considering the lifting of the casing, the resistance of the shoe and formation is outside the scope of this study.

The buoyancy calculation sheet (Table 3.1) and Figure 3.1 are included in this section.



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Area calculations are included in Section 4.

### Results

- The buoyancy is a negative 1.63 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 1.70 million lb, so the hanger will not lift
- The resultant of the pressure forces on the hanger is a downward force.
- The casing and hanger will not lift in this condition.
- The weight of the production casing is more than the weight of the displaced mud. Since this condition has the heaviest displaced weight of all the conditions, this indicates that the buoyant force will be positive (cause the casing to lift) only with a high differential between the annulus pressure and the production casing pressure at the hanger.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 9077 psi
  - Interior pressure 9077 psi
  - Vertical force 73500 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 13200 psi
  - Interior pressure 13200 psi
  - Vertical force 132000 lb, upwards

### Condition 2

#### *Basis*

The wellbore is filled with mud fluid. Communication through the annulus cement allows reservoir fluid into the annulus.

The production casing is filled with mud to the RKB.

The annulus is filled with mud to the production casing hanger and is sealed at the hanger. The pressure at the top of the annulus is determined by hydrostatic weight of the mud using the reservoir pressure as the known pressure. The pressure at the bottom of the annulus is reservoir pressure. Mud flows into the reservoir until pressures are equal.

Cement seals the reservoir from the production casing, so the pressure inside the production casing is the hydrostatic pressure of the mud.

Since the production casing is sealed at the bottom, lifting the casing will lift the contents of the casing up to the RKB, and so the weight of the contents is included in the buoyancy calculation. This weight of the contents will act to oppose the buoyancy.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.



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Friction between the production casing and cement or annular or interior fluids was not considered. The production casing does not lift in this condition so friction does not apply. The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing was ignored per the reasoning stated in Condition 1.

The buoyancy calculation sheet (Table 3.2) and Figure 3.2 are included in this section.

### Results

- The buoyancy is a negative 1.63 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 1.84 million lb, so the hanger will not lift
- The decrease in the net force on the hanger from Condition 1 is due to the reduction in annulus pressure at the hanger.
- The resultant of the pressure forces on the hanger is a downward force.
- The casing and hanger will not lift in this condition.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 8354 psi
  - Interior pressure 9077 psi
  - Vertical force 46078 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 13200 psi
  - Interior pressure 11844 psi
  - Vertical force 132000 lb, upwards

### Condition 3

#### *Basis*

Communication through the annulus cement allows reservoir fluid into the annulus. The annulus is filled with production fluid whose properties are determined from a HYSYS model for a static column. The composition of the production fluid was taken from the Schlumberger report (Appendix B, Item 6).

The production casing contains mud to the RKB.

This condition would occur if the reservoir – annulus communication occurred before the seawater was introduced into the production casing.

Cement seals the reservoir from the production casing, so the pressure inside the production casing is the hydrostatic pressure of the gel/seawater/mud column.

Since the production casing is sealed at the bottom, lifting the casing will lift the contents of the casing up to the RKB, and so the weight of the contents is included in the buoyancy calculation. This weight of the contents will act to oppose the buoyancy.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.



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The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7” casing.

Friction between the production casing and cement or annular or interior fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1

The buoyancy calculation sheet (Table 3.3), HYSYS model description and results, and Figure 3.3 are included in this section.

### *Results*

- The buoyancy is a negative 2.0 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 0.34 million lb, so the hanger will not lift
- The displaced fluid weight is reduced from Condition 1 because the production fluid is lighter than the mud.
- The annulus pressure at the hanger is higher than the interior pressure in the production casing, adding to the buoyant force.
- The net buoyancy force is greater than Condition 1.
- The casing and hanger will not lift in this condition.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 10540 psi
  - Interior pressure 9077 psi
  - Vertical force 128990 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 13200 psi
  - Interior pressure 11835 psi
  - Vertical force 132000 lb, upwards

### Condition 4

#### *Basis*

Communication through the annulus cement allows reservoir fluid into the annulus. The production casing contains a drill string from the RKB to 8367 feet TVD. The production casing contains seawater above 8367 feet through the wellhead to the riser. The riser is filled with seawater to 300 feet below the RKB. The top 300 feet contains a gel with a density of 16.3 ppg.

This condition would occur if the reservoir – annulus communication occurred after the seawater was introduced into the production casing.

The annulus is filled with production fluid with properties per Condition 3.



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Cement seals the reservoir from the production casing, so the pressure inside the production casing is the hydrostatic pressure of the mud.

Since the production casing is sealed at the bottom, lifting the casing will lift the contents of the casing up to the RKB, and so the weight of the contents is included in the buoyancy calculation. This weight of the contents will act to reduce buoyancy.

The drill string is supported by the rig and has no effect on the buoyancy of the production casing.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or (annular or interior) fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1

The buoyancy calculation sheet (Table 3.4), the HYSYS model description and results, and Figure 3.4 are included in this section.

#### *Results*

- The buoyancy is a negative 1.56 million lb, so the production casing will not lift
- The net force on the hanger is a negative 0.34 million lb, so the hanger will not lift.
- The weight of the production casing contents is reduced since the gel/seawater/mud column is lighter than that of 14 ppg mud.
- The net buoyancy force is greater than Condition 1.
- The annulus pressure at the hanger is greater than in Condition 1.
- The resultant of the pressure forces on the hanger is an upward force.
- The casing and hanger will not lift in this condition.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 10540 psi
  - Interior pressure 6818 psi
  - Vertical force 196363 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 13200 psi
  - Interior pressure 11835 psi
  - Vertical force 132000 lb, upwards



### **Condition 5**

#### *Basis*

Communication through the production casing cement allows reservoir fluid into the production casing. The production casing contains a drill string from the RKB to 8367 feet TVD. The production casing contains seawater above 8367 feet through the wellhead to the riser. The riser is filled with seawater to 300 feet below the RKB. The top 300 feet is a gel with a density of 16.3 ppg.

This condition would occur if the production casing – reservoir communication occurred after the seawater was introduced into the production casing.

The annulus is filled with mud from the cementing operations. Pressures in the annulus are the same as Condition 1.

The bottom of the production casing is at reservoir pressure. The pressure profile in the production casing is the hydrostatic head of the gel/seawater/mud combination. The hydrostatic pressure of the gel/seawater/mud column is 11083 psi, less than the reservoir pressure of 11844 psi. The fluid level in the production casing will rise as the higher pressure reservoir flows into the casing.

Since the production casing is not sealed at the bottom, lifting the casing will not lift the contents of the casing up to the RKB. Therefore the weight of the contents is excluded from the buoyancy calculation.

The drill string is supported by the rig and has no effect on the buoyancy of the production casing.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or annular or interior fluids was not considered because the value of friction force is dependent on many factors that are unknown. Friction, however, will only apply when the casing moves.

The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1.

The buoyancy calculation sheet (Table 3.5) and results and Figure 3.5 are included in this section.

The production casing and hanger may lift in this condition. When the hanger lifts, the pressures will equalize between annulus and the interior and the fluid in the annulus will flow to the interior. The amount of lift cannot be determined with the available information.



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### Results

- The production casing interior is underbalanced by about 756 psi, so the reservoir will flow into the casing.
- The buoyancy is a negative 0.11 million lb, so the production casing will not lift.
- The net force on the hanger is a positive 0.11 million lb, so the production casing can lift. Note the buoyancy force is included in this force.
- The casing and hanger can lift in this condition. Reservoir fluid has already entered the production casing in this condition, so the effect on flowrate of production fluids is small.
- The amount of lift was not determined.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 9078 psi
  - Interior pressure 6818 psi
  - Vertical force 140896 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 11835 psi
  - Interior pressure 11835 psi
  - Vertical force 118350 lb, upwards

### Condition 6

#### *Basis*

Communication through production casing cement allows reservoir fluid into both the annulus and the production casing. The production casing contains production fluid in a static column (a flowing well is outside the study scope due to the uncertainty of the flowrate). The annulus also contains a column of static production fluid.

This condition would occur if reservoir communication occurred and if the communication were established between both the annulus and the production casing.

Production fluid properties were determined as in Condition 3. The bottom of the annulus and the production string are at reservoir pressure.

The pressure profile in the production casing is the combination of hydrostatic head of the production fluid and the friction drop of the flowing fluid. The production fluid has either swept the previous gel/seawater/mud fluid out of the production casing or has flowed around it.

The rate of flow is unknown. Since the pressure in the production casing is used only to determine the downward force on the hanger, intermediate pressures and temperatures are not determined.

Since the production casing is not sealed at the bottom, lifting the casing will not lift the contents of the casing up to the RKB. Therefore the weight of the contents is excluded from the buoyancy calculation.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.



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The casing shoe was treated as 7" casing.

Friction between the production casing and cement or annular or interior fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1

The buoyancy calculation sheet (Table 3.6) and results and Figure 3.6 are included in this section.

### *Results*

- The buoyancy is a negative 0.47 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 0.64 lb, so the hanger will not lift.
- The resultant of the pressure forces on the hanger is a downward force. This is expected because the two pressures are equal and the interior area is larger than the annulus area.
- The bottom of the production casing is assumed to be open so the weight of the contents is not included in the buoyancy calculation.
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 10540 psi
  - Interior pressure 10540 psi
  - Vertical force 85346 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 11835 psi
  - Interior pressure 11835 psi
  - Vertical force 118350 lb, upwards

### Condition 7

#### *Basis*

Communication through production casing cement allows reservoir fluid into the production casing. The production casing contains production fluid in a static column (a flowing well is outside the study scope due to the uncertainty of the flowrate). The annulus contains a mud column from the original cement process.

This condition would occur if the reservoir communication occurred and if communication were established between the production casing but with no communication with the annulus.

Production fluid properties were determined as in Condition 3. The bottom of the production casing is at reservoir pressure. The pressures of the annulus are as in Condition 1.

The pressure profile in the production casing is the combination of hydrostatic head of the production fluid and the friction drop of the flowing fluid. The production fluid has either swept the previous gel/seawater/mud fluid out of the production casing or has flowed around it.

The rate of flow is unknown. Since the pressure in the production casing is used only to determine the downward force on the hanger, intermediate pressures and temperatures were not determined.





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Since the production casing is not sealed at the bottom, lifting the casing will not lift the contents of the casing up to the RKB. Therefore the weight of the contents is excluded in the buoyancy calculation.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or annular or interior fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1.

The buoyancy calculation sheet (Table 3.7) and results and Figure 3.7 are included in this section.

### *Results*

- The buoyancy is a negative 0.11 million lb, so the production casing will not lift.
- The net force on the hanger is a negative 1.28 million lb, so the hanger will not lift.
- The resultant of the pressure forces on the hanger is a downward force. This is because the interior pressure is higher (8816 psig) than the annulus pressure (3797 psig).
- The pressures and forces at the crossover due to the pressures and area differences are:
  - Annular pressure 9077 psi
  - Interior pressure 10540 psi
  - Vertical force 29872 lb, upwards
- The pressures and forces at the bottom of the casing are:
  - Annular pressure 11835 psi
  - Interior pressure 11835 psi
  - Vertical force 118350 lb, upwards

### Condition 8

#### *Basis*

Communication through production casing cement allows reservoir fluid into the production casing. The production casing contains production fluid in a static column (a flowing well is out of the study scope due to the uncertainty of the flowrate). The annulus contains a column of production fluid that is static.

This condition would occur if the reservoir communication occurred and if communication were established between the production casing but with no communication with the annulus.



## Section 3 – Calculations & Figures

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Production fluid properties for the annulus were determined as in Condition 3. The bottom of the production casing is at reservoir pressure. The pressures of the annulus are as in Condition 1.

The pressure profile in the production casing was set at 2358 psi at the TO casing and 11835 psi (reservoir pressure) at the bottom. No attempt was made to model the flowing well fluid and the pressure profile in the casing. The production fluid has either swept the previous gel/seawater/mud fluid out of the production casing or has flowed around it.

The rate of flow is unknown. Since the pressure in the production casing is used only to determine the downward force on the hanger, intermediate pressures and temperatures are not determined.

Since the production casing is not sealed at the bottom, lifting the casing will not lift the contents of the casing up to the RKB. Therefore the weight of the contents is excluded in the buoyancy calculation.

Pressures in the annulus and in the production casing affect the buoyant force on the hanger as described in Condition 1.

The areas of the production casing hanger exposed to the annulus and production casing pressures are as described in Condition 1.

The casing shoe was treated as 7" casing.

Friction between the production casing and cement or annular or interior fluids was not considered. The production casing does not lift in this condition so friction does not apply.

The ability of the cement to hold the production casing in place was not considered.

The ability of the shoe and bottom of the hole formation to resist the lowering of the production casing will be ignored per the reasoning stated in Condition 1.

The buoyancy calculation sheet (Table 3.8) and results and Figure 3.8 are included in this section.

The hanger will lift in this condition. When the hanger lifts, the pressures will equalize between annulus and the interior and the fluid in the annulus will flow to the interior. We could not determine the amount of lift with the current information.

### Results

- The buoyancy is a negative 0.47 million lb, so the production casing will not lift.
- The net force on the hanger is a positive 0.75 million lb, so the hanger can lift.
- The resultant of the pressure forces on the hanger is an upward force. This is because the interior pressure is lower (2358 psig) than the annulus pressure (8816 psig).
- The pressures and forces at the crossover due to the pressures and area differences were not determined.
- The pressures and forces at the bottom of the casing are:
  - Annular pressure      11835 psi
  - Interior pressure      11835 psi
  - Vertical force          118350 lb, upwards



Section 3 – Calculations & Figures

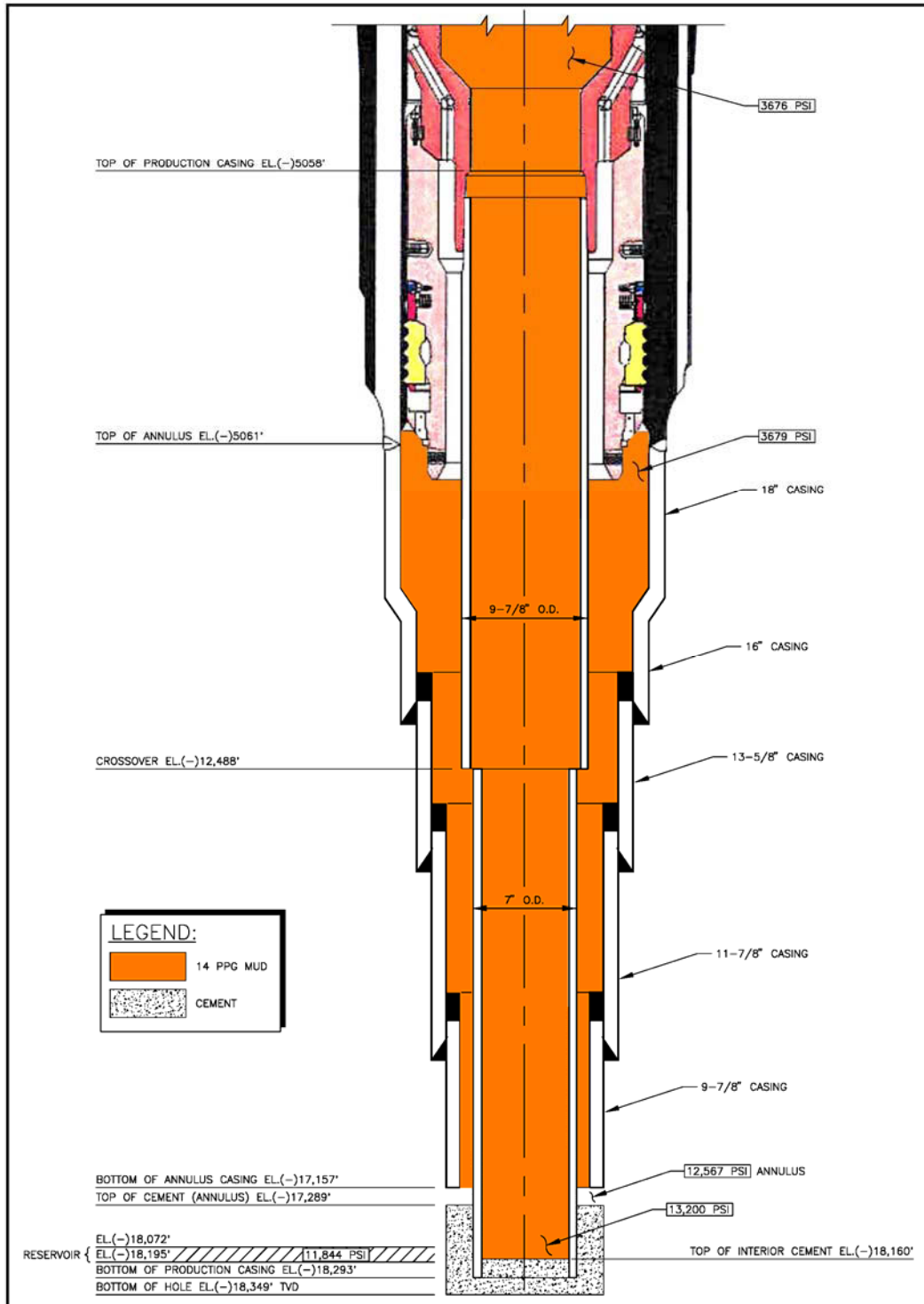


Figure 3.1 – Condition 1

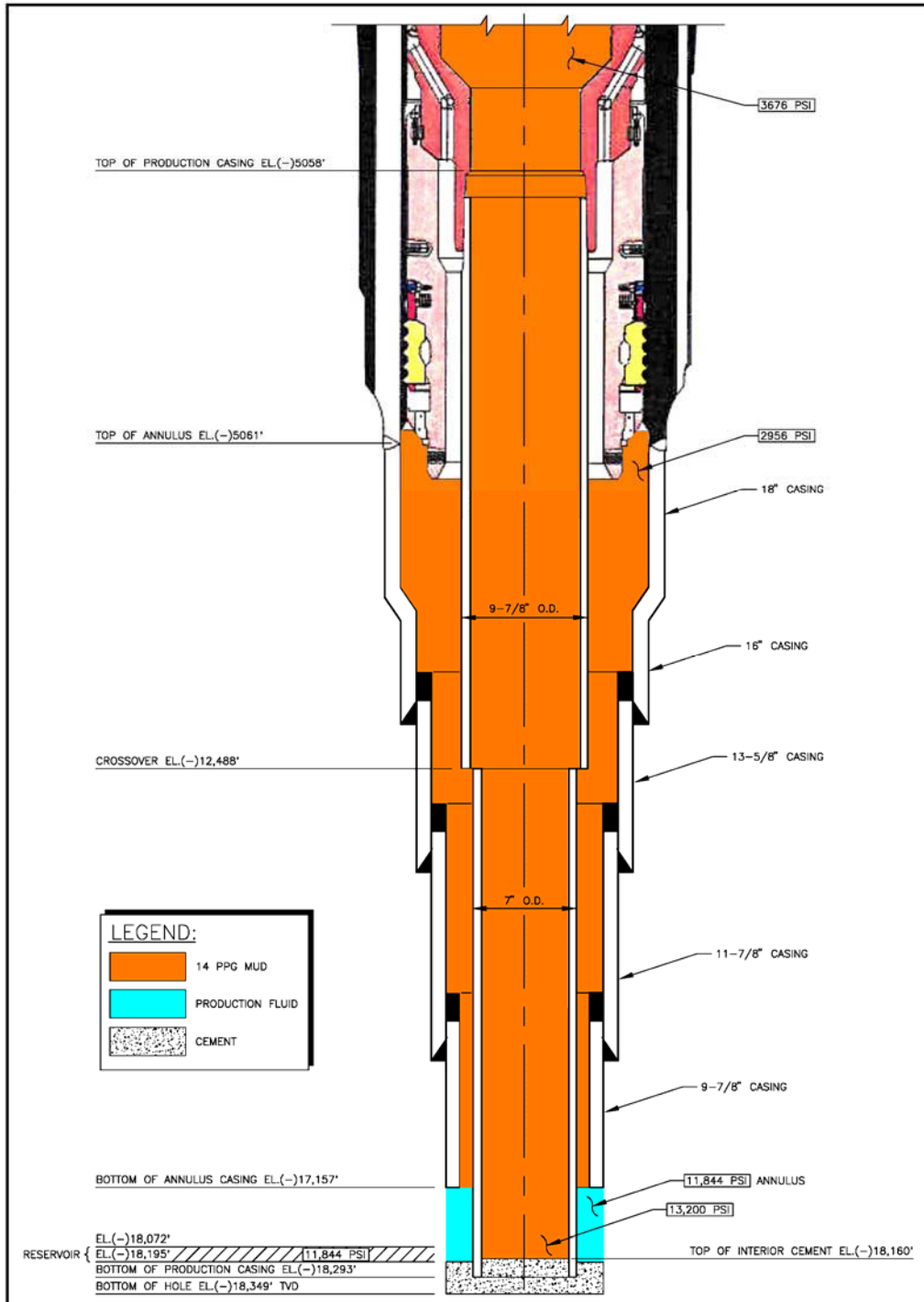


Figure 3.2 – Condition 2



Section 3 – Calculations & Figures

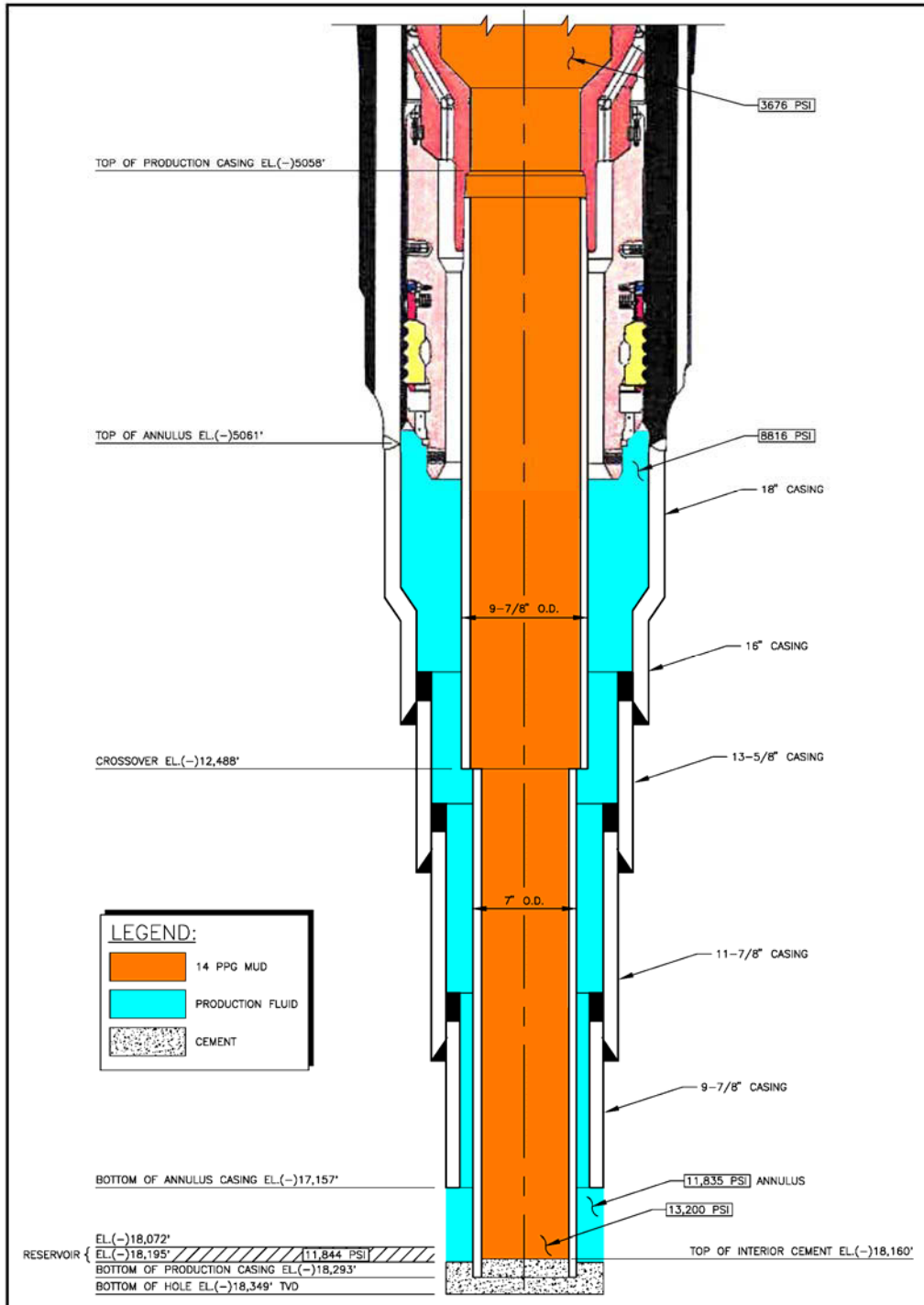


Figure 3.3 – Condition 3

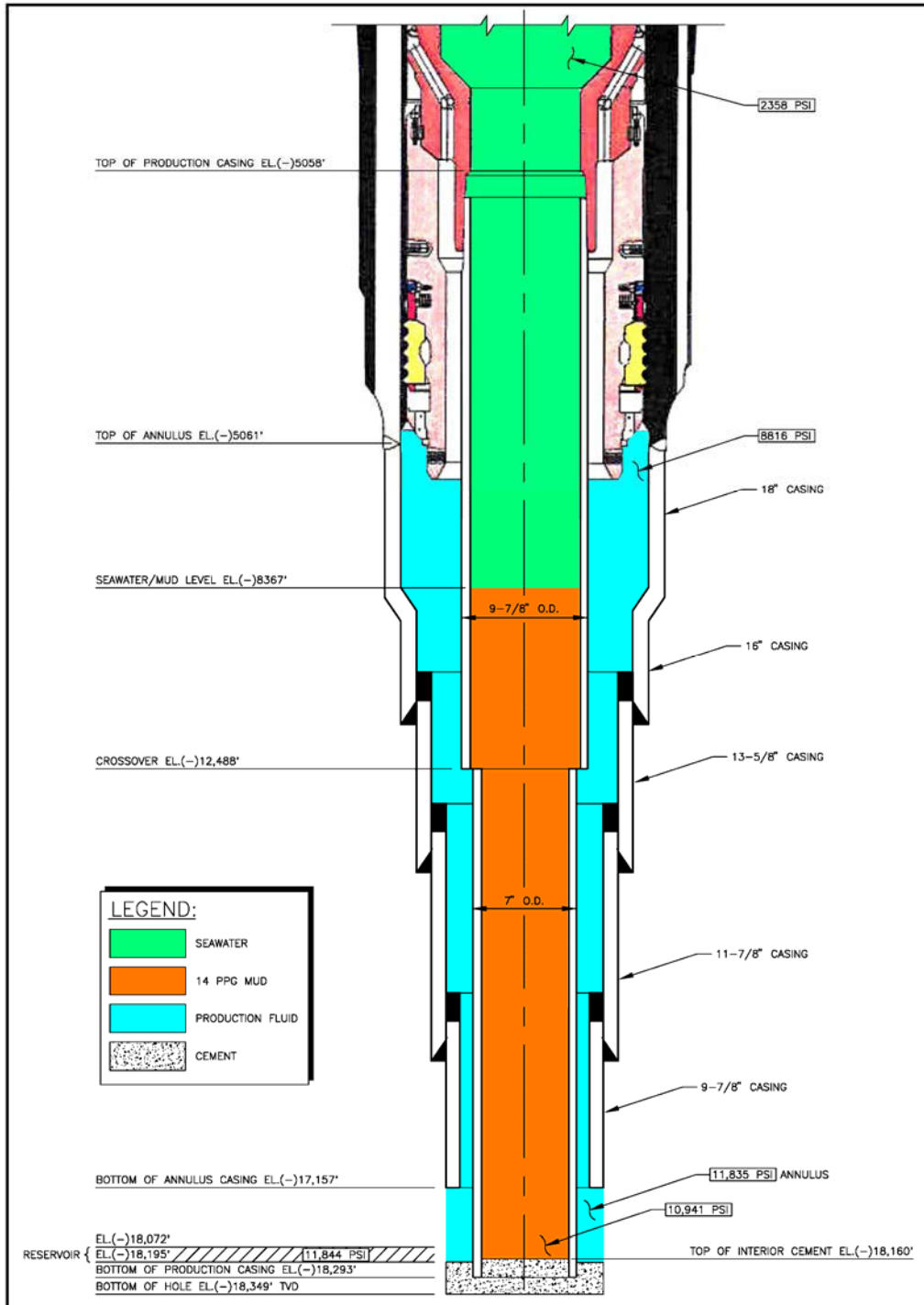


Figure 3.4 – Condition 4



Section 3 – Calculations & Figures

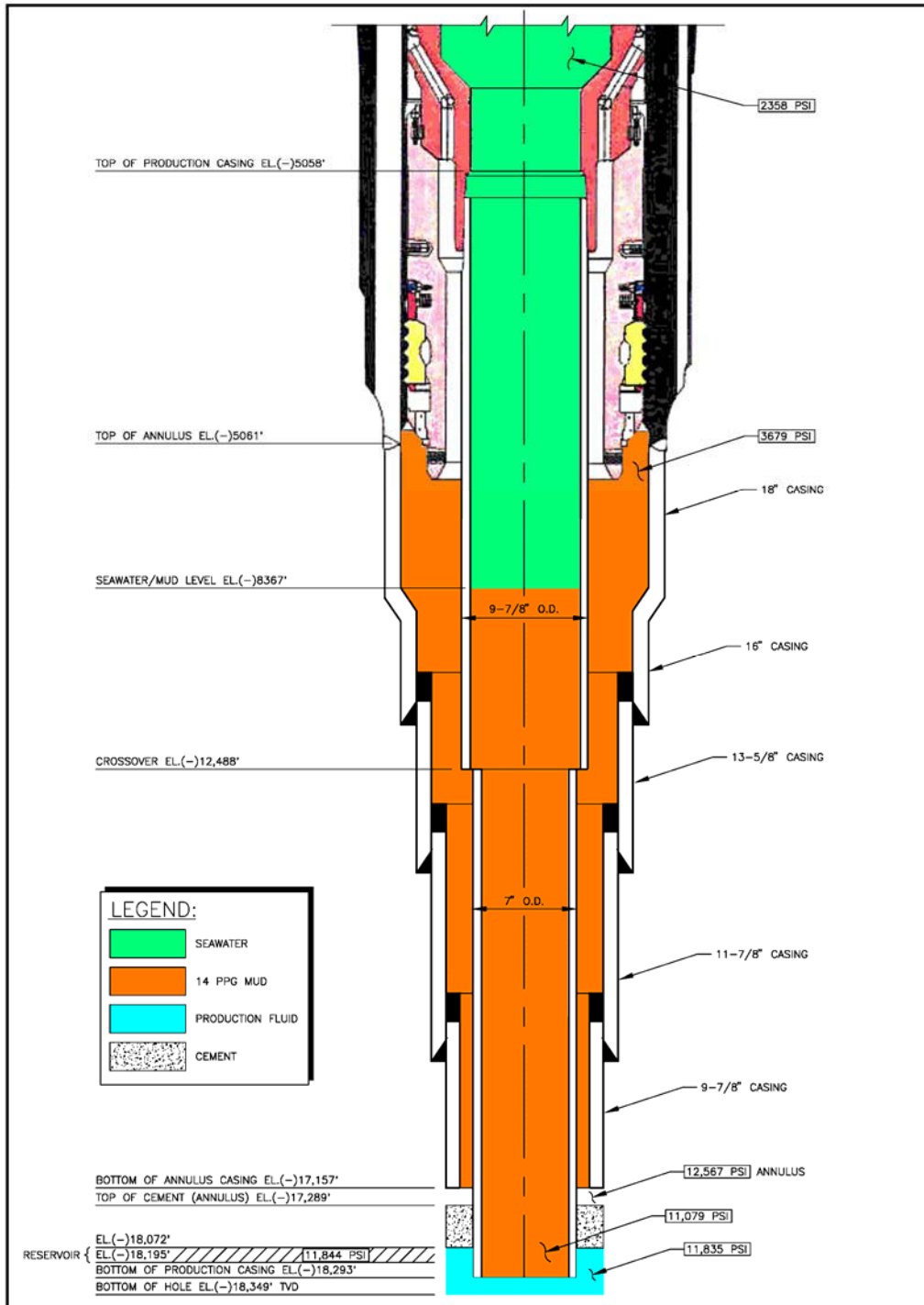


Figure 3.5 – Condition 5



Section 3 – Calculations & Figures

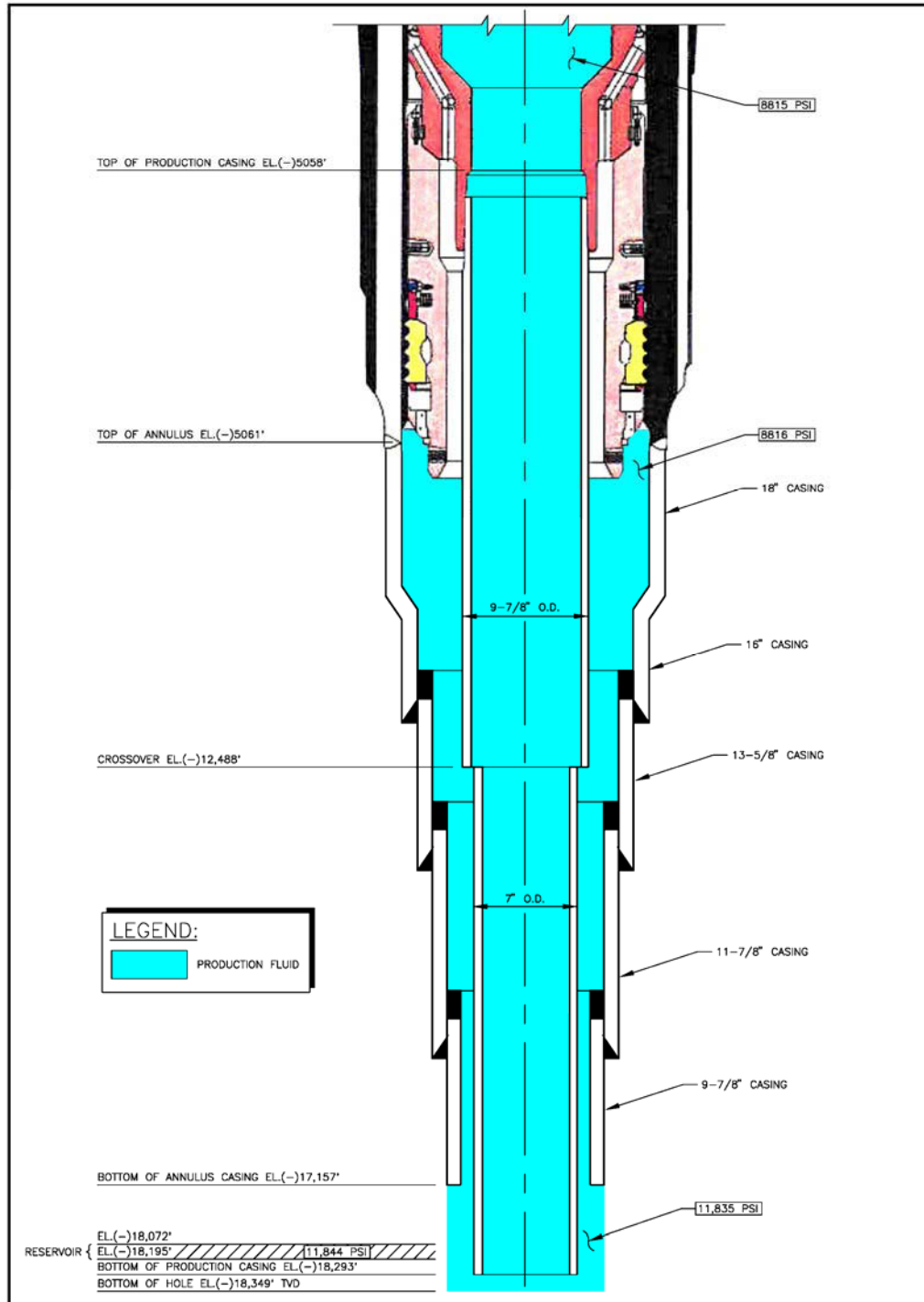


Figure 3.6 – Condition 6





Section 3 – Calculations & Figures

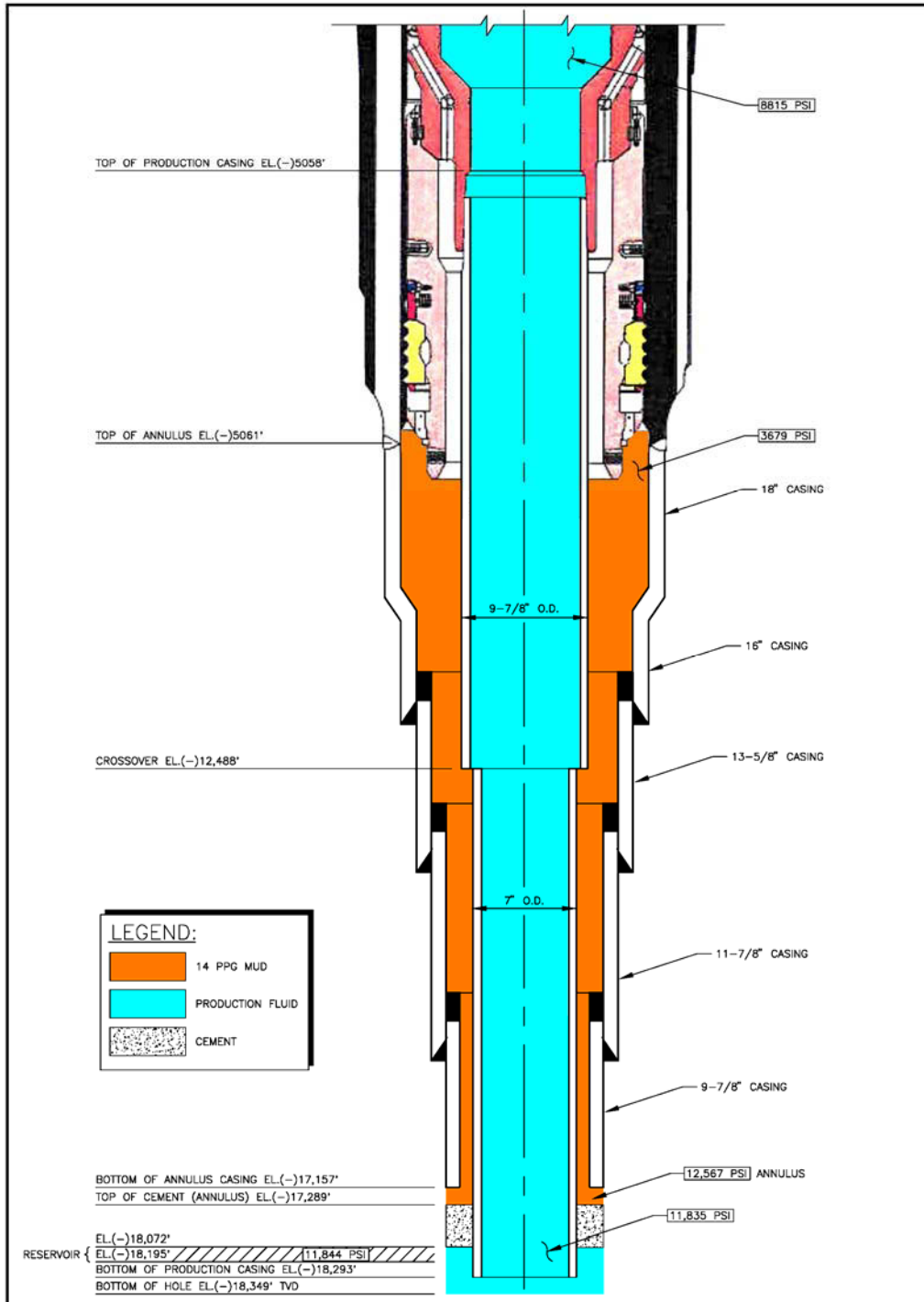


Figure 3.7 – Condition 7

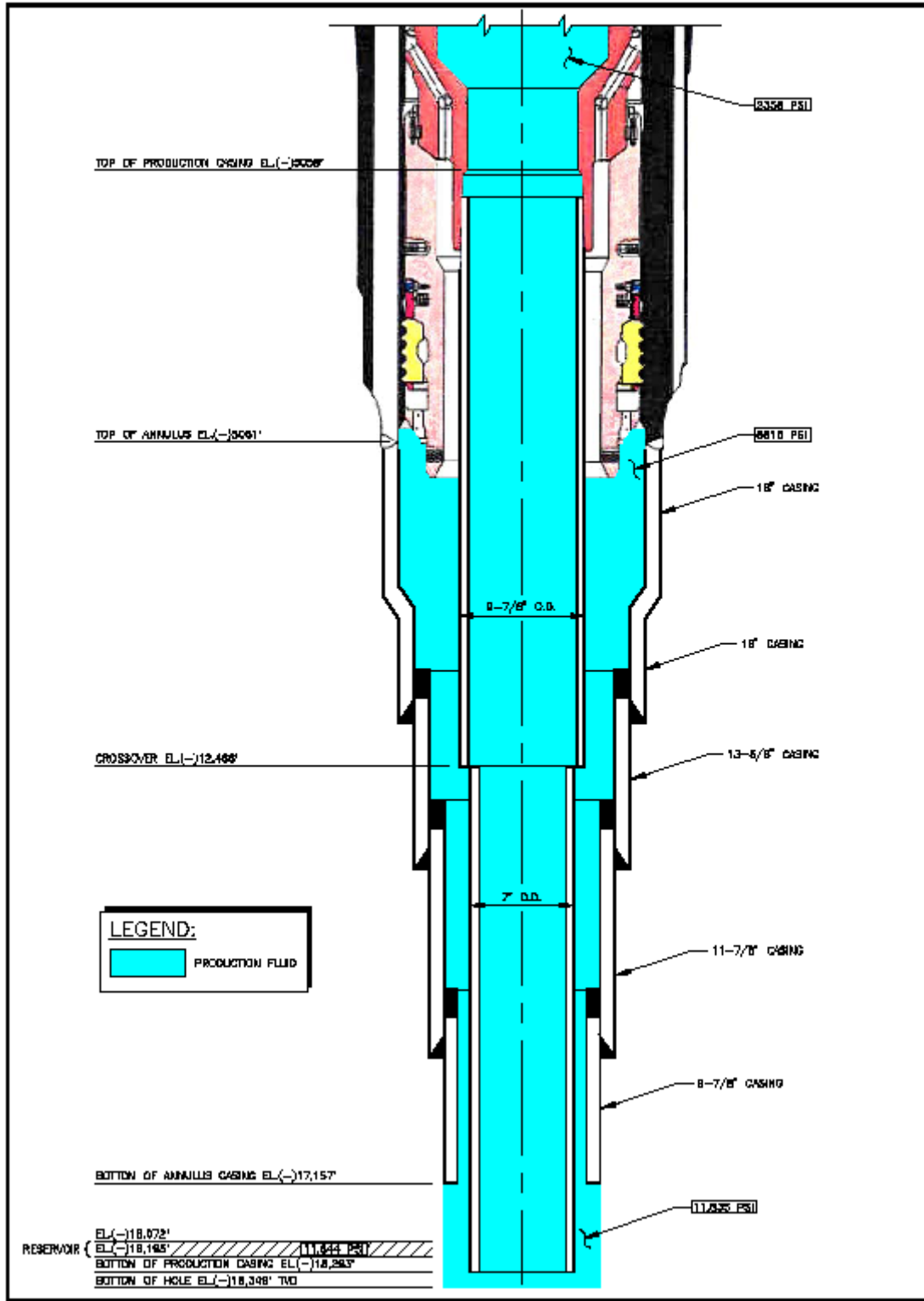


Figure 3.8 – Condition 8



Section 3 – Calculations & Figures

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Table with columns: Comments, Start Time, End Time, Production Description, Production Casing ID, Production Casing Fluid Temp, Production Casing Fluid Weight, Production Casing Segment, Production Casing Annulus Description, Annulus ID for Displacement, Annulus Fluid Temp, Annulus Fluid Weight, Annulus Pressure, Production Casing Fluid Weight, Production Casing Fluid Temp, Production Casing Fluid Weight, Production Casing Fluid Temp, Production Casing Fluid Weight.

Temperature Criteria: Above mudline, temperatures are in equilibrium with seawater per pycnocline curve. In wellbore, mud in annulus and production casing are in equilibrium with formations. Annulus Casing Temps from Downhole Temp surveys. Cement density: 5.26 bb/ premium H at 87.4 ppg. 38.9 bb/ Premium H, foamed to 47.75 bb/ 14.5 ppg. 6.03 bb/ Premium H 87.4 ppg. 20 bb/ spacer 14.3 ppg. 53 bb/ SOBIM 14.0 ppg. 728.5 bb/ SOBIM 14.0 ppg (69 ft. cement in shoe)

Cement seals reservoir. Cement plug prevents production fluid from flowing into Production Casing. M fills Production Casing. Since Production Casing is plugged, the contents of the Production Casing affect the buoyancy. 89 feet of cement in Production Casing. Area of P Production Casing seal = 195.71 sq in. Pressure at top of Annulus = 3676 psi. Upwards force = 78627 lb. Area of 'B' seal = 244.85 sq in. Pressure wellhead = 3676 psi. Downwards force = 788871 lb. Net Force on hanger, lbm = Displaced weight - casing weight upwards force - downwards force - weight of fluid in Production Casing = -170E+06 lb

Table 3.1 – Condition 1



Section 3 – Calculations & Figures

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Table with columns: Comments, Start True Vertical Depth, End True Vertical Depth, Production Casing, Production Casing Capacity, Production Casing Weight, Production Casing Description, Annulus ID for Displacement, Estor/P, Annulus Fluid Ave Temp, Deg F, Annulus Fluid Pressure at end, Annulus Fluid Pressure at start, Production Casing Displacement, Production Casing Displacement, Production Fluid Temp gradient, Annulus Fluid Temp gradient, Production Casing Material.

Notes:
Communication between the reservoir and the annulus, mud still in the annulus.
Cement plug prevents production fluid from flowing into Production Casing.
Mud fills annulus, cement does not hold Production Casing.
Mud fills Production Casing.
Since Production Casing is plugged, the contents of the Production Casing affect the buoyancy.
Pressure at top of Annulus set equal to balance reservoir pressure at bottom.
Area of Production Casing seal = 185.71 sq in
Pressure at top of Annulus = 2956 psi
Upwards force = 578454 lb
Area of 8" seal = 214,846 sq in
Pressure wellhead = 3676.45 psi
Downwards force = 789871 lb
Net Buoyancy, lbm = Displaced weight - casing weight upwards force - downwards force
+Weight of fluid in Production Casing = -184E+06
Net Buoyancy, lb

Table 3.2 – Condition 2



Section 3 – Calculations & Figures

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Table with columns: Comments, Start True Vertical Depth, End True Vertical Depth, Production Casing, Production Casing ID, Production Casing Ave Temp, Production Casing Fluid Temp, Production Casing Fluid Pressure, Production Casing Fluid Capacity, Production Casing Fluid Weight, Production Casing Fluid Description, Annulus ID for Displacement, Exterior Fluid, Annulus Fluid Ave Temp, Annulus Fluid Pressure, Annulus Fluid Density, Production Casing Displacement, Production Casing Fluid Gradient, Production Casing Material.

Notes: Cement fails allowing production fluid to flow into annulus. Production fluid fills annulus, is trapped by seals at T O Annulus. No flow of production fluid. Pressure at T O annulus is set by reservoir pressure and density of production fluid. Cement plug prevents production fluid from flowing into Production Casing. Cement does not fix Production Casing. Mud fills Production Casing. Area of Production Casing seal = 85.71 sq in. Pressure at top of Annulus = 8876 psi. Upwards force = 1725401 lb. Area of 8" seal = 214.85 sq in. Pressure wellhead = 3876.45 psi. Downwards force = 763871 lb. Net Buoyancy lbm = Displaced weight - casing weight upwards force - downwards force = +Weight of fluid in Production Casing - 106E+06. Net Buoyancy, lb

Table 3.3 – Condition 3



Section 3 – Calculations & Figures

Table with columns: Comments, Start True Vertical Depth, End True Vertical Depth, Production Casing Description, Production Casing ID, Production Casing Fluid, Production Casing Temp, Production Casing Ave Temp, Production Casing Capacity, Production Casing Weight, Annulus Description, Annulus Displacement, Annulus D/Tor Displacement, Annulus Fluid Ave Temp, Annulus Fluid Ppg, Annulus Pressure at end, Production Casing Fluid Displacement, Production Casing Temp, Product on Casing Temp, Product on Casing Material.

Notes:
Communication between the reservoir and the annulus.
Production fluid fills annulus & is trapped by seal at T O Annulus, no flow of production fluid
Pressure at T O Annulus is set by reservoir pressure and density of production fluid
Cement plug prevents production fluid from flowing into Production Casing.
Cement does not fix Production Casing.
Since Production Casing is plugged the contents of the P nodule Casing affect the buoyancy.
Area of Production Casing seal = 85.71 sq in
Pressure at top of Annulus = 886 psi
Upwards force = 1725401 lb
Area of 8" seal = 214.85 sq in
Pressure wellhead = 2358 psi
Downwards force = 506667 lb
Net Buoyancy, ltm = Displaced weight - casing weight+upwards force - downwards force
=Weight of fluid in Production casing - 3.39E+05
Net Buoyancy, lb

Table 3.4 – Condition 4



Section 3 – Calculations & Figures

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Table 3.5 - Condition 5. Columns include: Comments, Start True Vertical Depth, End True Vertical Depth, Production Casing ID, Production Casing Ave Temp, Production Casing Ave PPG, Production Casing Pressure at end of segment, Production Casing Capacity, Production Casing Weight, Annulus Casing Description, Annulus ID for Displacement, Exterior Fluid, Annulus Fluid Ave Temp, Annulus Fluid Ppg, Annulus Pressure at end, Production Displacement, Production Casing Fluid Displacement, Production Casing Temp Gradient, Annulus Fluid Temp Gradient, Production Casing Material.

Notes:
Communication between reservoir and Production Casing. Gel/seawater/mud still in Production Casing.
Mud in annulus, cement does not hold Production Casing.
Production Casing contains seawater down to depth of the drill pipe.
Since holes are open, the contents of the Production Casing do not affect the buoyancy.
Area of Production Casing seal = 85.71 sq in
Pressure at top of Annulus = 3679 psi
Upwards force = 720081 lb
Area of 'B' seal = 24.85 sq in
Pressure wellhead = 2356 psi
Downwards force = 506687 lb
Net Buoyancy, lbm = Displaced casing weight upwards force - downwards force
+ Weight of fluid in Production Casing = 407E+05
Net Buoyancy, lb = 2567 psi

Table 3.5 – Condition 5



Section 3 – Calculations & Figures

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Table with multiple columns: Comments, Start True Vertical Depth, End True Vertical Depth, Production Casing Description, Production Casing ID, Production Casing Fluid Temp, Production Casing Ave Temp, Production Casing Capacity, Production Casing Weight, Production Casing Weight per Segment, Annulus Description, Annulus ID for Displacement, Annulus Fluid Ave Temp, Annulus Pressure at End, Production Casing Displacement, Production Casing Fluid Temp, Annulus Pressure at Segment, Production Casing Material.

Notes: Communication between reservoir and Production Casing. Production fluid densities, pressures, and temperatures for annulus from HYSYS calculation for static column. File name: DensityCalc10Deg.F

Pressure at T O annulus is set by reservoir pressure and density of production fluid. Production fluid is in annulus, cement does not hold P production Casing.

Since hole is open, the contents of the Production Casing do not affect the buoyancy. Area of Production Casing seal = 95.7 sq in

Upwards force = 8896 psi

Area of 'B' seal = 1725401 lb

Pressure wallhead = 8896 psi

Downwards force = 8693934 lb

Net Buoyancy, lbm = Displaced weight - casing weight-upwards force - downwards force

+Weight of fluid in P production Casing = -6.4 E+05

Table 3.6 – Condition 6





Section 3 – Calculations & Figures

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Table with columns: Comments, Start True Vertical Depth, End True Vertical Depth, Production Casing Description, Production Casing ID, Production Casing Fluid, Production Casing Fluid Temp, Production Casing Ave Temp, Production Casing Fluid DegF, Production Casing Capacity, Production Casing Weight, Production Casing Weight per segment, Annulus Casing Description, Annulus ID for Displacement, Annulus ID for Fluid, Annulus Fluid Ave Temp DegF, Annulus Fluid DegF, Annulus Pressure at end, Production Casing Fluid Displacement, Production Casing Fluid Displacement per ft, Production Casing Fluid Temp gradient, Production Casing Material.

Notes:
Communication between reservoir and Production Casing.
Mud fills annulus, is trapped by seals at T.O Annulus, no flow of production fluid.
Pressure at T.O annulus is set by reservoir pressure and density of mud.
Production fluid is in annulus, cement does not hold Production Casing.
Production Casing contains production fluid.
Since hole is open, the contents of the Production Casing do not affect the buoyancy.
Area of Production Casing seal = 95.71 sq in
Upwards force = 3679 psi
Area of B\* seal = 24.8 sq in
Downwards force = 88.5 psi
Net Buoyancy (lbm) = Displaced weight - casing weight upwards force - downwards force
+Weight of fluid in Production Casing =
Net Buoyancy (lb) -1.28E+06

Table 3.7 – Condition 7



Section 3 – Calculations & Figures

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Table with columns: Comments, Start/End True Vertical Depth, Production Casing Description, Production Casing ID, Production Casing Fluid, Production Casing Ave Temp, Production Casing Ppg, Production Casing Pressure at end of segment, Production Casing Capacity, Production Casing Weight, Production Casing Description, Annulus ID or Displacement, Annulus Fluid, Annulus Pressure at end, Production Casing Displacement, Production Casing Displacement w/gradient, DM Press at end of segment, Production Casing Temp, Annulus Temp, Production Casing Material.

Notes:
Communication between reservoir and Production Casing.
Production fluid fills annulus... is trapped by seals at T O Annulus... no flow of production fluid...
Pressure at T O Annulus is set by reservoir pressure and density of production fluid.
Production fluids in annulus... cement does not hold P production Casing.
Since hole is open, the contents of the Production Casing do not affect the buoyancy.
Area of Production Casing seal = 185.7 sq in
Pressure at top of Annulus = 886 psi
Upwards force = 1725401 lb
Area of 18" seal = 214.85 sq in
Pressure wellhead = 2356 psi
Downwards force = 508667 lb
Net Buoyancy, lbm = Displaced weight - casing weight upwards force - downwards force
Weight of fluid in Production Casing = 746E+05
Net Buoyancy, lb

Table 3.8 – Condition 8



# Section 4 - Basis of Calculations

## Temperature Calculations

For temperature growth

$$\Delta L = L \cdot \alpha \cdot \Delta T$$

Where:

L = characteristic length

alpha = thermal coefficient of material, in/in per Deg F

delta T = change in Temperature, Deg F

For 9-7/8" casing, close to the crossover:

I D, in, = 8.625

b, in, = 4.3125

t, in, = 0.625

At 12486 feet depth,

T, temperature at depth, Deg F = 178

T0, temperature at floor, Deg F = 70

delta T, Deg F = 108

alpha = 6.26E-06

b, inner id, in, = 4.3125

a, outer id, in, = 4.9375

Delta a, in, = a\*alpha\*delta T, in

Delta a, in, = 3.34E-03 or 0.0676 percent, negligible

Delta b, in, = b\*alpha\*delta T, in

Delta b, in, = 2.91E-03 or 0.0676 percent, negligible

The increase in steel volume is accompanied by a reduction in density.

For 7" casing, close to the shoe:

O D, in, = 7

b, in, = 3.047

t, in, = 0.453 R/t = 6.726269

At 18349 TVD feet depth,

T, temperature at depth, Deg F = 262

T0, temperature at floor, Deg F = 70

delta T, Deg F = 192

alpha = 6.52E-06

b, inner id, in, = 3.047

a, outer id, in, = 3.5

Delta a, in, = a\*alpha\*delta T, in

Delta a, in, = 4.38E-03 or 0.1253 percent, negligible

Delta b, in, = b\*alpha\*delta T, in

Delta b, in, = 3.82E-03 or 0.1253 percent, negligible

The increase in steel volume is accompanied by a reduction in density.



**Section 4 – Basis of Calculations**

**Pressure Calculations**

per Roark & Young, "Formulas for Stress and Strain, Fifth Edition:

Table 29, "Formulas for membrane stresses and deformations in thin walled pressure vessels ", pp 448

for vessel with radius R, thickness t under uniform load q (psi)  
applicable for R/t > 10

$$\Delta R = q \cdot R^2 / (E \cdot t)$$

E = modulus of elasticity, psi

For 9-7/8" casing, close to the crossover:

I D, in, =	8.625		
R, in, =	4.3125		
t, in, =	0.625	R/t =	6.90

For 7" casing, close to the shoe:

O D, in, =	7		
R, in, =	3.047		
t, in, =	0.453	R/t =	6.73

Do not use, R/t is too small.

Table 39, "Formulas for thick walled vessels under internal and external loading", pp 505

For 9-7/8" casing, the maximum differential is at the hanger in condition 4

I D, in, =	8.625		
R, in, =	4.3125		
t, in, =	0.625	R/t =	6.9

At 5061 foot depth, 48 Deg F

inner pressure, psi, =	2358	(Condition 4)	
Outer pressure , psi, =	8816		
net inner pressure, psi, =	-6458	Condition 2	
E, psi, =	2.99E+07	Temp =	at 70 Deg F
mu, Poisson's ratio =	0.3		

b, inner id, in, =	4.3125	L, length, in =	89160
a, outer id, in, =	4.9375		

Delta a, in, = $q/E \cdot 2 \cdot a \cdot b^2 / (a^2 - b^2)$		
Delta a, in, =	-0.008986	or -0.182 percent, negligible
Delta b, in, = $q \cdot b / E \cdot ((a^2 + b^2) / (a^2 - b^2) + \mu)$		
Delta b, in, =	-0.007197	or -0.167 percent, negligible
Delta L, in, = $-q \cdot \mu \cdot L / E \cdot 2 \cdot b^2 / (a^2 - b^2)$		
	37.134928	or 0.042 percent, negligible



**Section 4 – Basis of Calculations**

**Pressure Calculations (Cont'd)**

For 7" casing, the maximum differential is at the bottom hole in condition 5

O D, in, = 7

R, in, = 3.047

t, in, = 0.453 R/t = 6.726269

At 17163 feet depth,

inner pressure, psi, = 11079

Outer pressure , psi, = 12576

net inner pressure, psi, = -1497

E, psi, = 2.92E+07

Temp = at 262 Deg F

mu, Poisson's ratio = 0.3

b, inner id, in, = 3.047

L, length, in = 68064

a, outer id, in, = 3.50

Delta a, in, =  $q/E*2*a*b^2/(a^2-b^2)$

Delta a, in, = -0.001483 or -0.042 percent, negligible

Delta b, in, =  $q*b/E*((a^2+b^2)/(a^2-b^2)+mu)$

Delta b, in, = -0.001181 or -0.039 percent, negligible

Delta L, in, =  $-q*mu*L/E*2*b^2/(a^2-b^2)$

27.59 or 0.041 percent, negligible



**Section 4 – Basis of Calculations**

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**Differential Pressure**

		Casing size, in		Differential, psi	
		9-7/8"	7"		
Condition 1					
hanger	Pi, psig	3676			
	Po, psig	3676		0	
xovr	Pi, psig	9077			
	Po, psig	9077		0	
BH	Pi, psig		12567		
	Po, psig		13200	-633	
Condition 2		9-7/8"	7"		
hanger	Pi, psig	3676			
	Po, psig	2956		-720	
xovr	Pi, psig	9077			
	Po, psig	8358		719	
BH	Pi, psig		13200		
	Po, psig		11844	1356	
Condition 3		9-7/8"	7"		
hanger	Pi, psig	3676			
	Po, psig	8816		-5140	
xovr	Pi, psig	9077			
	Po, psig	10540		-1463	
BH	Pi, psig		13200		
	Po, psig		11844	1356	
Condition 4		9-7/8"	7"		
hanger	Pi, psig	2358			
	Po, psig	8816		-6458	max
xovr	Pi, psig	6818			
	Po, psig	10540		-3722	max
BH	Pi, psig		10941		
	Po, psig		11835	-894	
Condition 5		9-7/8"	7"		
hanger	Pi, psig	2358			
	Po, psig	3679		-1321	
xovr	Pi, psig	6818			
	Po, psig	9078		-2260	
BH	Pi, psig		11079		
	Po, psig		12567	-1488	max
Condition 6		9-7/8"	7"		
hanger	Pi, psig	8815			
	Po, psig	8816		-1	
xovr	Pi, psig	10540			
	Po, psig	10540		0	
BH	Pi, psig		11835		
	Po, psig		11835	0	



**Section 4 – Basis of Calculations**

**Differential Pressure (Cont'd)**

		Casing size, in		Differential, psi
		9-7/8"	7"	
Condition 7				
hanger	Pi, psig	8815		
	Po, psig	3679		5136
xovr	Pi, psig	10540		
	Po, psig	9077		-1463
BH	Pi, psig		11835	
	Po, psig		11835	0
Condition 8				
hanger	Pi, psig	2358		
	Po, psig	8816		-6458
xovr	Pi, psig	NA		
	Po, psig	NA		
BH	Pi, psig		11835	
	Po, psig		11835	0

**Material Properties**

Low Chrome Moly steels, Cr less than 3%

Modulus of elasticity from ANSI B31.3

Temp, Deg F	E, psi (multiply table by 10 <sup>6</sup> )		
-100	30.4		
70	29.9		
200	29.5	178	29.6
300	29.0	262	29.19
262	29.19		
162	29.6		

Mean Coefficient of thermal expansion between 70 Deg F and temperature, from ANSI B31.3

Temp, Deg F	A, in/in (multiply table by 10 <sup>-6</sup> )		
70	0		
125	6.19		
150	6.25		
175	6.31	178	6.26
200	6.43		
250	6.49		
275	6.54		
262	6.52		
162	6.28		



**Section 4 – Basis of Calculations**

**Volumes and weights of Displaced Fluids**

19.5" ID conductor =	.3694 bbl/ft =	15.419 gal/ft
18.375 " ID casing =	.3280 bbl/ft =	13.776 gal/ft
14.92" ID casing =	.2162 bbl/ft =	9.080 gal/ft
9-7/8" OD casing =	.09473 bbl/ft displacement =	3.979 gal/ft
9-7/8" OD, 8.626 " ID casing =	.07227 bb//ft =	3.035 gal/ft
7" OD casing, displacement =	.0476 bbl/ft =	1.999 gal/ft
7" OD, 6" ID casing =	0.036071 bbl/ft =	1.515 gal/ft

9-7/8" OD, 62.8 #/ft casing is 8.626 "  
 7" OD, 32 #/ft casing is 6" ID  
 16", 97#/ft casing is 14.92" ID

"Prod. Casing Fluid ppg" used density of seawater at 80 Deg F  
 Fresh water = 8.318 lb/gal at 80 Deg F (Ref. No. 9)  
 Seawater sp gr = 1.025  
 Seawater = 8.526 lb/gal

Production Fluid density is determined from HYSYS runs which give static gradient at each depth.  
 gradient (in/ft) / 12 = gradient (psi/ft)      psi/ft \* 8.337 / .433 = lb/gal.  
 7.38257 inH2O/ft= 0.26475 psi/ft = 5.098 lb/gal

Density of seawater at depths is read from the graph of density vs. sea depth, (Ref. No.2)  
 This data is generic, site specific data was not used.

Density of mud was fixed at 14 ppg. No data was given for the change in density with pressure or temperature.

"Prod Casing Pressure at end of segment" is determined from:  
 Fluid ppg / 8.37 (freshwater ppg) \* .433 (gradient of freshwater) \* length of segment plus previous pressure

"Prod Casing Fluid Capacity" is determined from:  
 Either id of casing or tables (Ref. No. 2)

Temperatures of fluid and casing were determined by:  
 For pumped seawater, temperature is 80 Deg F.  
 For mud in Production Casing, temperature is in equilibrium with the formations.  
 Temperatures of the formations were determined from Ref. No. 1  
 Temperatures of produced fluids were determined from Appendix B, Item 6.

Reservoir pressure was determined from Halliburton, Appendix B Item 5.

Annulus Casing Description is not applicable for the segments of the riser, BOP and the Production Casing hanger.

These items are supported by the well head and surface conductor and do not affect the loads on the Production Casing. The contents of the riser, BOP and Production Casing hanger would move if the Production Casing moves.

Therefore, the contents of the riser, BOP and Production Casing hanger are included in the Production Casing loads.





Section 4 – Basis of Calculations

Crossover Dimensions

Crossover dimensions from K&B drawing. (See Apeendix B, Item.4)

- 1 Cross over is 10.078 " od, 8.626" id for 8.5"
- 2 Cross over is 10.078 " od, 6.034" id at 48-32.161-8.5 = 7.34"
- 1.5 Tapers from 1 to 2 over the 7.34"
- 3 Cross over is 10.078 " od, 6.034" id for 32.382-48+32.151"=16.533"
- 4 Cross over is 7.016" od, 6.034" id at 6.9"
- 3.5 Tapers from 3 to 4 over 48-32.392-6.9 = 8.708"

7.339

16.533

8.708

so:	10.087 " od by	8.626 " id for	8.5 "	volume of OD =	679.26 cu in
				volume of ID =	496.74 cu in
Taper is	10.087 " od by	7.33 " id for	7.34 "	volume of OD =	586.56 cu in
				volume of ID =	309.74 cu in
	10.087 " od by	6.034 " id for	16.533 "	volume of OD =	1321.19 cu in
				volume of ID =	472.77 cu in
Taper is	8.5515 " od by	6.034 " id for	8.708 "	volume of OD =	500.14 cu in
				volume of ID =	249.01 cu in
	7.016 " od by	6.034 " id for	6.9 "	volume of OD =	266.76 cu in
				volume of ID =	197.31 cu in

0.2875 cu ft = 2.15 gal

0.9990 cu ft = 7.47 gal

1.94 cu ft = 14.51 gal

0.999 cu ft = 7.47 gal

470.59 lb

Displacement is total of OD volume = 3353.90 cu in =

Contents is total of ID volume = 1725.57 cu in =

Weight is difference of OD volume and ID volume = 1628.33 cu in =

Density is 8000 kg/cu m = 0.289 lb/cu in

7700 0.287

8000 0.289

8030 0.29

Check totals

Add section 1-4

47.981 Overall length from drawing = 48 inches

Length is ok.



## Section 4 – Basis of Calculations

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### Seal Area Calculations

Pressure, force downward

Pressure = pressure in wellhead = pressure at top of Production casing = pressure at 5058 foot depth

Area = inside of 18.620" Seal = Area (18.62" dia ) - Area (8.553" dia)

214.8461 sq in

Pressure, force upwards

Pressure = pressure in annulus at T O Annulus = pressure at 5061 foot depth

Area = inside of 18.620" Seal = Area (18.62" dia ) - Area (9.875" dia)

195.7124

### Forces at Crossover

Crossover dimensions from K&B drawing, (See Appendix B, Item.4)

1 Cross over is 10.078 " od, 8.626" id at upper end

4 Cross over is 7.016" od, 6.034" id at lower end

External area

The external vertical force on the crossover is the external pressure times the horizontal projection of the outside area

The internal vertical force on the crossover is the internal pressure times the horizontal projection of the inside area

Outside area, below reducer = 10.087 OD by 7.016" ID = A<sub>o</sub> = 54.14 sq in

Outside area, above reducer = 10.087 OD by 9.875" ID = A<sub>o</sub> = 16.21 sq in

Inside area = 8.625" OD by 6.034" ID = A<sub>i</sub> = 29.83 sq in

Note the two outside area counteract each other.

Pressure forces

Pressure forces are due to the different pressures and areas at the crossover.

Radial forces.

The Horizontal forces are due to the difference in the external and internal pressures.

The pressures are listed below because the radial pressures are used to determine stress levels in the casing.

Condition	External Pressure, psi	Internal Pressure, psi
1	9077	9077
2	8354	9077
3	10540	9077
4	10540	6818
5	9078	6818
6	10540	10540
7	9077	10540



## Section 4 – Basis of Calculations

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### Forces at Crossover (Cont'd)

#### Vertical Forces

The external vertical force on the crossover is the external pressure times the horizontal projection of (the below outside area - the above outside area.)

The internal vertical force on the crossover is the internal pressure times the horizontal projection of the inside area

The forces oppose each other because the external vertical force is an upward force, and the internal force is a downwards force.

For simplicity, the same external pressure is used for the above and below outside areas.

These areas are only 4 feet apart.

Condition	External Pressure, psi	Internal Pressure, psi	Net upwards vertical force, lbm
1	9077	9077	73500
2	8354	9077	46078
3	10540	9077	128988
4	10540	6818	196363
5	9078	6818	140896
6	10540	10540	85346
7	9077	10540	29872

These forces are included in the buoyancy calculations by Archimedes principle.



## Section 5 - HYSYS Calculations

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### WELL SIMULATION USING ASPEN HYSYS (ver. 7.1)

The well bore sample report from Schlumberger (Appendix B) was reviewed for the best fit for the component input requirements for HYSYS. The sample chosen was sample 1.18. The component information supplied was entered into the computer for simulation. Components heavier than C30 were combined to form a C30+ component. Since the oil/gas composition is eighty-five mole percent gas, the C30+ component will not significantly change the simulation.

Once the input stream compositions were entered and a physical property database chosen, the process flow for the well simulation was modeled in HYSYS. The sample component results were entered for the gas and the oil phases at a set reservoir pressure of 11,856 psia and reservoir temperature of 243 degrees Fahrenheit. A mixer was used to combine the phases into one stream at a 11,856 psia and 243 degrees Fahrenheit, then sent through a heat exchanger to adjust temperature and pressure to the sample collection conditions of 236 degrees Fahrenheit and 11,835.3 psig. The output was connected to a pipe with a length and vertical elevation specified to match the bottom hole to RKB TVD. The exit temperature was also specified. The velocity in the pipe was minimized in order to approximate a static column of fluid. The chosen output numbers were density, temperature and pressure at various elevations. These outputs were inputs into the buoyancy calculation worksheets.

The specified output temperature was chosen to be 110 deg F at rig floor elevation. This approximates a normal wellhead temperature for this type of production.

In this simulation, the pipe length and elevation (well inlet is zero length at zero elevation) were set to 18,074 feet. The molar flow was set to 100 moles per hour to make checking of the component composition percentages easier and to minimize the flow through the pipe (i.e. Well) to approach static conditions.

The well profile output was extracted and loaded into an Excel spreadsheet. The density and pressure were used in the buoyancy spreadsheets.

To check sensitivity to temperature, the specified output temperature was changed to 160 Deg F at rig floor elevation. All other parameters were held constant. The output for the 110 Deg F outlet temperature case was compared to that for the 160 outlet temperature case. The pressure differences at the production casing hanger were different by only 0.36%. For the buoyancy calculations, this difference is negligible, so the pressures are insensitive to the chosen outlet temperature.

The temperature of the static column will drop with time due to lower temperatures in the formations eventually reaching the formation temperatures. This process will take an undetermined length of time. The densities of the fluid will be higher so the hydrostatic pressures will be lower at the top of the column. (The bottom pressure is controlled by the reservoir pressure.) The higher temperatures used in the model therefore result in higher pressures at the hanger. The results in the buoyancy calculations are an upper bound for pressures at the hanger.

The buoyant force that lifts the casing in Condition 4 will be lower as the production fluid cools.



## Section 6 - List of References

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1. "Density of Ocean Water", Pycnocline graph Windows to the Universe website, accessed 8/24/2010
2. Schlumberger\i-Handbook
3. Crane "Flow of Fluids" Technical Paper No. 410 Physical Properties of Water, pg-A6
4. Roark & Young "Formulas for Stress & Strain", Fifth Edition, 1975 McGraw-Hill ppg 504, ppg 583



## Appendix A – List of Abbreviations

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KEI	Keystone Engineering Inc
BOEMRE	Bureau of Ocean Energy, Management, Regulation and Enforcement
RKB	Rig Kelly Bushing
ppg	pound per gallon
SOBM	Synthetic oil based mud
T O	Top Of
BH	Bottom Hole
BOP	Blow Out Preventer
TVD	True Vertical Depth
sq in	Square Inch
psi	Pounds per square inch
lb	Pounds
wt	Weight
x-over	crossover
Deg F	Degrees Fahrenheit
ID	inner diameter
OD	outer diameter
bbl	barrel (42 gallons)



## Appendix B – Material Supplied by BOEMRE

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1. Wellbore Schematic, File Name "BP-HZN-MBI000184559.xls" BP GoM Deepwater Exploration "Macondo" As-Drilled Schematic, Rev. 15, 4/22/2010
2. Halliburton "BP AMERICA PRODUCTION COMPANY PO Box 22024 Tulsa OK 74121-2024 Macondo #1" 9 7/8" x 7" Production Casing Design Report" for Brian Morel, April 18, 2010
3. Dril-Quip BP Proposal Drawing for Wellhead System, Part Number 2-PD-32424-02CP "Macando" , File name DQ-DHJI-000011
4. K & B Machine Works Drawing of Crossover – 9 7/8" 62.80# TSH 523 BOX x 7" 32.0# TSH 513 PIN x 48" LG, Drawing # KB-TOX-0978-0014, Rev. B, 3/30/10
5. Halliburton, Houston Technology Center, "GeoTap Pressure Transient Analysis, BP Exploration & Production Inc. Well: OCS-G 32306 001 ST00BP01..." by Steve Kizziar, 04/08/10
6. Schlumberger "Fluid Analysis on "Macondo" Samples, BP ...Well: OCS-G 32306 #1, Reservoir Sample Analysis Report", Updated 6/09/10 . Prepared by S.George Mathews, Schlumberger OilPhase – DBR